



Triadic orthogonal decomposition reveals nonlinearity in fluid flows

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Energy transfer across scales is fundamental in fluid dynamics, linking large-scale flow motions to small-scale turbulent structures in engineering and natural environments. Triadic interactions among three wave components form complex networks across scales, challenging understanding and model reduction. We introduce triadic orthogonal decomposition (TOD), a method that identifies coherent flow structures optimally capturing spectral momentum transfer, quantifies their coupling and energy exchange in an energy-budget bispectrum and reveals the regions where they interact. Triadic orthogonal decomposition distinguishes three components – a momentum recipient, donor and catalyst – and recovers laws governing pairwise, six-triad and global triad conservation. We apply TOD to three examples: the classical cylinder wake, experimental wind turbine wake data and a direct numerical simulation of isotropic turbulence. Energy transfer can be spatially distributed but vanish upon integration or spatially localised but facilitate net interscale exchange, so a complete characterisation of nonlinearity requires examination of both integral and local transfers. In the cylinder wake, we link backscatter of energy from high to low frequencies to a compact attenuation region downstream of the cylinder. In the turbine wake, we confirm the known association between energy amplification and decay and vortex tilting, but observe more complex secondary mechanisms in suboptimal modes. For isotropic turbulence, we derive and confirm inertial-range frequency scaling for convective–recipient covariances, then demonstrate self-similar energy transfer at each rank.

Key words: nonlinear instability, Navier–Stokes equations, low-dimensional models

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1. Introduction

Spectral tones are a common feature of both natural and engineering flows, providing valuable insights into the extrinsic and intrinsic, as well as linear and nonlinear, dynamics of fluid systems. These tones appear across a wide range of applications, from engineering systems like turbines and propellers to geophysical flows shaped by planetary motion. While some arise from direct forcing, others emerge intrinsically through hydrodynamic instabilities or resonances. Identifying and distinguishing these spectral features is essential for understanding flow physics, diagnosing instability mechanisms and developing reduced-order models for complex systems.

Fundamental tones either arise from extraneous forcing, or originate from intrinsic flow instabilities or mechanisms. External frequencies are prevalent in many technical applications, such as the blade-passing frequency in rotating machinery like turbines, fans and propellers (Kerwin 1986; Hansen & Butterfield 1993; Conlisk 1997; Sandberg & Michelassi 2022). In geophysical and atmospheric flows, tonal peaks corresponding to semi-diurnal, diurnal and annual frequencies are driven by Earth's rotation, axial tilt and orbital dynamics relative to the Sun and Moon (Baldwin *et al.* 2001; Dijkstra & Burgers 2002; McWilliams 2016; Timmermann *et al.* 2018). Hydrodynamic instabilities, such as Kelvin–Helmholtz waves in shear-layer flows, represent the second source of fundamental tones. These tones can often be predicted by hydrodynamic stability theory as discrete eigenvalues of the linearised dynamics (Schmid & Henningson 2001; Drazin & Reid 2004). However, even linearly stable flows can exhibit tones when forced near an eigenfrequency, a phenomenon known as linear resonance. Extraneously forced and stable, but non-normal, systems may also undergo pseudo-resonance (Trefethen *et al.* 1993).

Nonlinear tones, on the other hand, arise from the interactions of fundamental or other nonlinear tones. In fluid flows, these originate from the quadratic nonlinearity of the convective term in the Navier–Stokes equations, which generates triadic sum- and difference-frequency and wavenumber interactions (Phillips 1960; Hammack & Henderson 1993). Resonant triadic interactions produce peaks at sub-, super- and ultra-harmonic frequencies. They occur in flows with time scales dominated by wave rather than eddy dynamics (Chen *et al.* 2005; Clark di Leoni & Mininni 2016), and efficiently couple widely disparate time and spatial scales. It is important to distinguish them from non-resonant triadic interactions consisting of compatible frequency or wavenumber triplets that nevertheless do not satisfy the resonant condition. The latter only redistribute energy without producing distinct tones, as seen in the broadband spectrum of isotropic, homogeneous turbulence (Pope 2000). While fundamental tones are often straightforward to identify by matching frequencies to known external sources or predictions from linear theory, nonlinear tones are typically identified by matching non-fundamental peaks in power spectra to integer multiples or sums and differences of fundamental frequencies. Since the power spectrum defines energy as the squared magnitude of the signal, it is inherently phase blind. Thus, peaks at sub- and super-harmonic frequencies are a necessary, but not sufficient, condition for confirming nonlinearity (Kim & Powers 1978). The occurrence of ultra-harmonic tones from triadic interactions between different spatial symmetry components adds further complexity to interpreting power spectra alone. The detection of significant nonlinear coupling is therefore predicated on an appropriate measure of the strength of triadic interactions. We will define such a measure, based on statistical correlation, for the analysis of spatio-temporal data.

A well-established tool for identifying triadic interactions in pointwise signals is the bispectrum, along with its normalised counterpart, the bicoherence. The bispectrum is a third-order statistic that correlates two frequency components with their sum, making it useful for detecting triadic frequency interactions that satisfy the zero-sum condition

by analysing the phase correlation among the three involved frequencies (see Brillinger (1965) for an early review). Building on this, Schmidt (2020) proposed bispectral mode decomposition (BMD), which uses an integral measure of bispectral density to identify coherent flow structures associated with triadic interactions. Bispectral mode decomposition achieves this by maximising the integral of the local product of two frequency components and their sum, obtained as the numerical radius of a spectral estimate of the bispectral density matrix. It identifies a single mode pair representing the dominant triad at each point in the bispectrum plane. The method introduced in this work, which we term triadic orthogonal decomposition (TOD), instead maximises the covariance between spectral components, and provides complete, orthogonal bases of convective modes and recipient modes, along with a modal energy budget that quantifies scale-to-scale energy transfer.

While the nonlinear term in the momentum equations can transfer and redistribute kinetic energy, it can neither produce nor remove net energy. The conservation of scale-to-scale kinetic energy transfer has been known at least as far back as the early text of Batchelor (1953), and was later expounded on by Kraichnan (1959) in the wavenumber domain. A systematic analysis of spectral energy conservation and transfer was also carried out by Smyth (1992), which is the foundation for this work. The properties of inter-triad energy conservation are also treated in standard texts, including Lesieur (1990), Schmid & Henningson (2001) and others. These properties are directly analogous in the frequency domain. Recent applications of kinetic energy transfer analysis and inter-triad conservation are found in the works of Barthel (2022) and Freeman, Martinuzzi & Hemmati (2024). The transport of kinetic energy in fluid flows is governed by the dynamical equations for the mean kinetic energy (MKE) and turbulent kinetic energy (TKE). These equations may be partitioned into terms that can be interpreted as distinct mechanisms for energy flow, see e.g. Pope (2000). They include: production, dissipation, advection and the transfer of energy through pressure, viscous diffusion and nonlinear interactions. Depending on the flow, each term represents either a contribution of energy to, or a removal of energy from, the overall kinetic energy budget. In Fourier space, the spectral kinetic energy equation describes the transfer of energy across scales, from frequency to frequency, and wavenumber to wavenumber. One of the pioneering examples of spectral energy-budget analysis of inhomogeneous flows is the work by Lumley (1964), which investigated the transfer of energy between wavenumbers in wall-bounded turbulence. It proposed the idea of an inverse energy cascade in which energy is transported from smaller to larger scales – a phenomenon now known to be general (Biferale, Musacchio & Toschi 2012). Smyth (1992) later proposed a formalism that enables a systematic examination of the directivity of energy transfer from a donor scale to a recipient scale. Similar observations about energy directivity were also made by Batchelor (1953). More recently, the donor–recipient interpretation has been used to study energy transfer in turbulence by Domaradzki & Rogallo (1990), Webber, Handler & Sirovich (2002), Alexakis *et al.* (2005a), de Wit *et al.* (2022) and Ding, Chung & Illingworth (2025), as well as in magnetohydrodynamic turbulence by Dar, Verma & Eswaran (2001) and Alexakis *et al.* (2005b), among others.

Whereas early studies of energy budgets tended to emphasise energy transfer as a local (pointwise) phenomenon, Rempfer & Fasel (1994) demonstrated that it is possible to incorporate the concept of coherent structures into a framework of modal energy-budget analysis. A comprehensive summary of the methodology and application to a complex bluff-body flow can be found in Hosseini, Martinuzzi & Noack (2016). In these works, coherent structures were deduced via space-only proper orthogonal decomposition (POD) and employed as the basis for a Galerkin reduced-order model (ROM) of the global energy

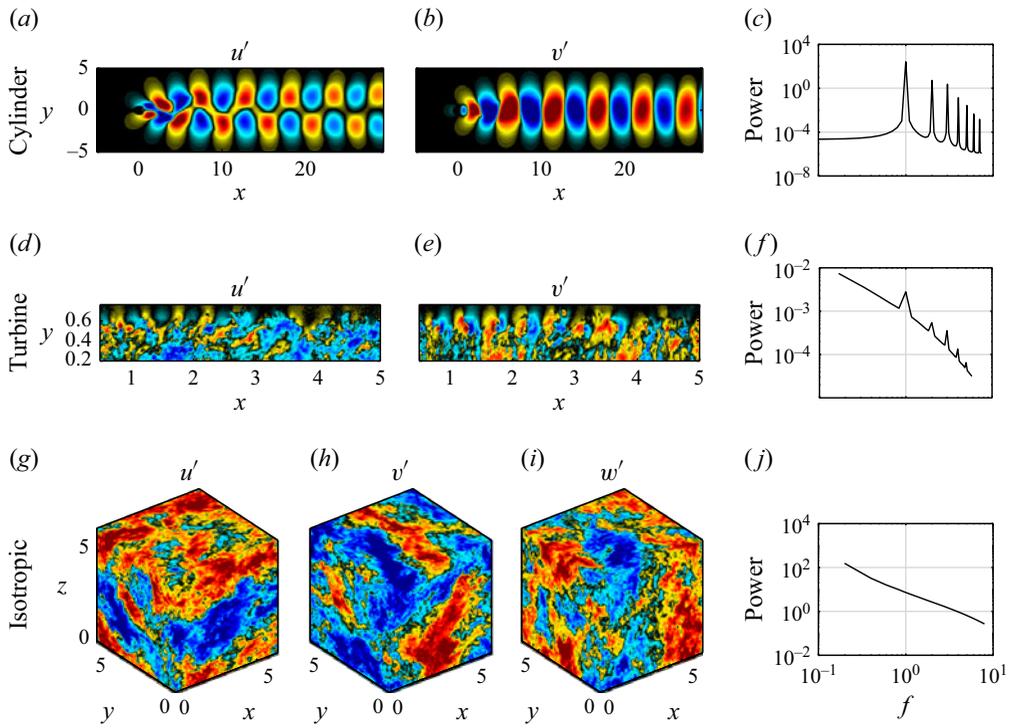


Figure 1. Instantaneous fluctuating velocities of the three examples considered in § 3 in the x (a,d,g), y (b,e,h) and z (i) directions: (a,b) cylinder wake; (d,e) wind turbine wake; ($g-i$) forced isotropic turbulence. Normalised velocity fields (blue, black, red, $q/|\max\{q\}| \in [-0.5, 0.5]$, with q representing any velocity component) are shown. Area- or volume-integrated power spectra are shown for the cylinder (c), wind turbine (f) and isotropic turbulence (j). In panels (d,e), for clarity the x - and y -axes are not drawn to scale. Unless otherwise noted, in what follows, all contours use the same axes limits and colour scales as (a,b) for the cylinder wake, (d,e) for the turbine wake or ($g-i$) for the isotropic turbulence.

dynamics. The model thus emphasises the transfer of energy mediated by interactions between structures. Instead of using POD modes, Jin, Symon & Illingworth (2021) used resolvent modes to compute the convective terms and energy transfers. These modal bases optimally represent the observable dynamics (in the case of POD) or the linear input–output system (in the case of resolvent analysis), with optimality defined in an energy norm. In contrast, TOD by construction explicitly targets individual nonlinear interactions by optimising the covariance between the convective and recipient mode pair for each triad. This feature also makes TOD ideally suited to the analysis of modal energy budgets in the frequency domain. Among the terms in the MKE and TKE budget equations, production, advection and nonlinear transfer are triadic in nature. We show that these triadic energy budgets can be directly recovered from the convective modes and recipient modes of the proposed decomposition.

It is arguably most natural to apply TOD to tonal flows, examples of which were given earlier. Their statistics exhibits a form of sparsity in which the nonlinearity is driven by a subset of all permissible triadic interactions. Indeed, our desire for a systematic approach to studying tonal flows provided the original impetus for this work. The first two applications we present, the wakes of a cylinder and a wind turbine, fall into this category. Their flow fields and power spectra are displayed in the first two rows of figure 1. From visual inspection it is obvious that there are discrete sets of dominant length and time

scales, which lock on to each other through the agency of the nonlinear term. Nonetheless, most fluid flows, particularly turbulent ones, are not spectrally sparse. For these flows it is intractable to enumerate all triadic interactions. An archetype of such broadband flows is Kolmogorov turbulence, which provides our third application. Its continuous spectrum is shown in the third row of [figure 1](#). Any description of its dynamics is necessarily incomplete unless all frequencies are considered. Compared with tonal flows, analysing the nonlinearity of broadband flows is thus an acute challenge. However, it is precisely when the number of interactions is large that it becomes imperative to discover patterns in the nonlinearity, whether for physical understanding or model reduction. As our third application demonstrates, for stochastic, broadband flows, TOD yields a ranked decomposition in which each rank in isolation carries clear significance in the context of known physical laws that govern nonlinear interactions – a perhaps surprising finding. The success of TOD in the analysis of both classes of flows, tonal and broadband, as well as laminar and turbulent, testifies to its potential and broad applicability.

The remainder of this paper is organised as follows. First, [§ 2.1](#) introduces the governing equations for linear momentum and kinetic energy in the frequency domain and the new orthogonal decomposition. Thereafter, [§ 2.2](#) covers the modal energy flow analysis, starting with the theory of the spectral energy budget in [§ 2.2.1](#) and inter-triad energy conservation in [§ 2.2.2](#), before applying these concepts to modal decomposition in [§ 2.2.3](#). Three applications of the decomposition are demonstrated in [§ 3](#): the cylinder wake at a Reynolds number of 100 as a canonical example of a nonlinear laminar flow in [§ 3.1](#), particle image velocimetry data of a wind turbine wake as an example of a turbulent engineering flow in [§ 3.2](#) and three-dimensional numerical simulation data of forced isotropic turbulence as an example of broadband flow in [§ 3.3](#). Finally, [§ 4](#) discusses our findings and summarises the method.

2. Methods

2.1. Triadic orthogonal decomposition

Throughout this work, we will use the following notational convention: variables which represent scalar-valued functions are italicised, e.g. q ; vector-valued functions are italicised and boldface, e.g. \mathbf{q} ; variables which are spatially discretised vectors are upright and boldface, e.g. \mathbf{q} . We denote the complex conjugate, transpose and Hermitian transpose by the superscripts $(\cdot)^*$, $(\cdot)^T$ and $(\cdot)^H$, respectively. [Figure 2](#) provides a bird’s eye view of the TOD algorithm, which is explained in detail in the following.

2.1.1. Spectral momentum and kinetic energy equations

The motion of an incompressible Newtonian fluid is governed by the momentum and continuity equations

$$\frac{\partial \mathbf{u}}{\partial t} = -(\mathbf{u} \cdot \nabla) \mathbf{u} - \nabla p + \frac{1}{Re} \nabla^2 \mathbf{u}, \quad (2.1a)$$

$$\nabla \cdot \mathbf{u} = 0, \quad (2.1b)$$

where $\mathbf{u} = [u \ v \ w]^T$ is the velocity vector field and p is the pressure field. Equation (2.1a) has been non-dimensionalised by the velocity scale, U_∞ , the length scale, L and is parameterised by the Reynolds number, $Re = U_\infty L / \nu$, where ν is the kinematic viscosity. Assuming a time-periodic flow, the flow variables can be expanded as the Fourier series

$$\mathbf{u}(\mathbf{x}, t) = \sum_{n=-\infty}^{\infty} \hat{\mathbf{u}}_n(\mathbf{x}) e^{i2\pi f_n t} \quad \text{and} \quad p(\mathbf{x}, t) = \sum_{n=-\infty}^{\infty} \hat{p}_n(\mathbf{x}) e^{i2\pi f_n t}. \quad (2.2)$$

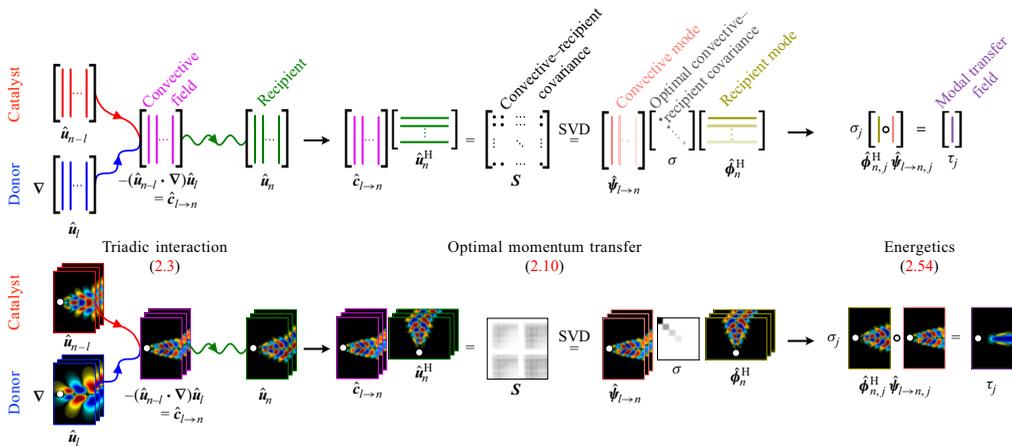


Figure 2. The TOD algorithm. The top row provides a graphical representation of the matrix algebra, while the bottom row illustrates the algorithm using a cylinder wake dataset. In this example, the convective and recipient modes are similar to the convective and recipient fields because the cylinder wake is laminar and periodic. Key equations are referenced by number. SVD stands for the singular value decomposition.

For each frequency, f_n , we obtain the frequency-domain representation of (2.1a)

$$i2\pi f_n \hat{\mathbf{u}}_n = - \left(\widehat{(\mathbf{u} \cdot \nabla) \mathbf{u}} \right)_n - \nabla \hat{p}_n + \frac{1}{Re} \nabla^2 \hat{\mathbf{u}}_n = - \sum_{l=-\infty}^{\infty} \underbrace{(\hat{\mathbf{u}}_{n-l} \cdot \nabla) \hat{\mathbf{u}}_l}_{-\hat{\mathbf{c}}_{l \rightarrow n}} - \nabla \hat{p}_n + \frac{1}{Re} \nabla^2 \hat{\mathbf{u}}_n, \tag{2.3}$$

where, in the second step, we have invoked the convolution theorem. This frequency-domain, or spectral, momentum equation lies at the core of our modal momentum transfer analysis. The special role and implications of the components in the convolution sum, $\hat{\mathbf{c}}_{l \rightarrow n}$, are discussed in § 2.1.2.

For the often-used notation of triads, (k, l, n) with $k = n - l$, any two of the frequency components determine the remaining one. For simplicity, we use the pairwise notation (l, n) to denote the triplet $(n - l, l, n)$ in the following. For the case of $n = 0$, (2.3) simplifies to

$$\begin{aligned} 0 &= -(\bar{\mathbf{u}} \cdot \nabla) \bar{\mathbf{u}} - \sum_{l \neq 0} \underbrace{(\hat{\mathbf{u}}_{-l} \cdot \nabla) \hat{\mathbf{u}}_l}_{-\hat{\mathbf{c}}_{l \rightarrow 0}} - \nabla \bar{p} + \frac{1}{Re} \nabla^2 \bar{\mathbf{u}} \\ &= -(\bar{\mathbf{u}} \cdot \nabla) \bar{\mathbf{u}} - \overline{(\mathbf{u}' \cdot \nabla) \mathbf{u}'} - \nabla \bar{p} + \frac{1}{Re} \nabla^2 \bar{\mathbf{u}}, \end{aligned} \tag{2.4}$$

which are the Reynolds-averaged Navier–Stokes (RANS) equations, with $\mathbf{u}' \equiv \mathbf{u} - \bar{\mathbf{u}}$. The definition of \mathbf{u}' implies that $\hat{\mathbf{u}}_l = \widehat{\mathbf{u}}'_l$ for $l \neq 0$, which leads to the second equality in (2.4). Each term in the RANS equations contributes to the deformation of the mean flow, $\bar{\mathbf{u}}$. The term $\hat{\mathbf{c}}_{l \rightarrow 0}$, in particular, is responsible for momentum transfer via the Reynolds stress, $\overline{(\mathbf{u}' \cdot \nabla) \mathbf{u}'}$.

Upon left-multiplying with $\hat{\mathbf{u}}_n^H$, (2.3) can be rewritten as the spectral kinetic energy equation

$$i2\pi f_n \hat{\mathbf{u}}_n^H \hat{\mathbf{u}}_n = -\hat{\mathbf{u}}_n^H \sum_{l=-\infty}^{\infty} \underbrace{(\hat{\mathbf{u}}_{n-l} \cdot \nabla) \hat{\mathbf{u}}_l}_{-\hat{\mathbf{c}}_{l \rightarrow n}} - \nabla \cdot (\hat{p}_n \hat{\mathbf{u}}_n^H) + \frac{1}{Re} \hat{\mathbf{u}}_n^H \nabla^2 \hat{\mathbf{u}}_n. \tag{2.5}$$

The spectral kinetic energy equation governs the distribution of the kinetic energy, $\hat{E}_n = 1/2(\hat{\mathbf{u}}_n^H \hat{\mathbf{u}}_n)$, and forms the basis for the modal energy flow analysis in § 2.2. A remarkable property of (2.5) regarding the convective term, which is not used here but discussed in detail by Freeman *et al.* (2024), is that, under certain conditions, the imaginary part of the term can be used to recover the integral power of the velocity, and the real part the integral dissipation.

A wide variety of open flows exhibit strong convective instability. Examples encompass natural and technical flows, such as bluff-body wakes and boundary layers, as well as exogenously forced flows like plasma-actuated jets. In these flows, the convective term $\hat{\mathbf{c}}_{l \rightarrow n}$ becomes dominant in both spectral momentum and kinetic energy equations. In the following, we outline how the spatial structures involved in spectral momentum transfer can be leveraged to identify and understand triadic flow interactions.

2.1.2. Recipient–donor framework

The recipient–donor framework (Batchelor 1953; Smyth 1992) of the momentum equations is a key concept that facilitates the calculation of both the direction and magnitude of modal energy transfer. In (2.3), we introduced the notation

$$\hat{\mathbf{c}}_{l \rightarrow n} = -(\hat{\mathbf{u}}_{n-l} \cdot \nabla) \hat{\mathbf{u}}_l \quad (2.6)$$

to represent the component in the convolution sum associated with the frequency pair (l, n) , or the corresponding triplet $(n - l, l, n)$. This notation implicitly denotes the direction of momentum transfer, $l \rightarrow n$, from frequency component $\hat{\mathbf{u}}_l$ to $\hat{\mathbf{u}}_n$. As (2.3) governs the velocity component at frequency f_n , we denote the component $\hat{\mathbf{u}}_n$ as the recipient of linear momentum (although contributions can be negative, resulting in a net loss of momentum). We first present an intuitive argument based on the roles of $\hat{\mathbf{u}}_{n-l}$ and $\hat{\mathbf{u}}_l$ in $\hat{\mathbf{c}}_{l \rightarrow n}$ in the familiar derivation of the momentum equations, which indicates that the velocity component $\hat{\mathbf{u}}_{n-l}$ advects $\hat{\mathbf{u}}_l$ in the direction of, and with a magnitude proportional to, its gradient field, $\nabla \hat{\mathbf{u}}_l$. Thus $\hat{\mathbf{u}}_l$ serves as a donor of linear momentum to $\hat{\mathbf{u}}_n$, while $\hat{\mathbf{u}}_{n-l}$ acts as the passive advector. To avoid confusion with the convective field, $\hat{\mathbf{c}}_{l \rightarrow n}$, which ultimately contributes to $\hat{\mathbf{u}}_n$ and hence frequency f_n , we refer to $\hat{\mathbf{u}}_{n-l}$ as the catalyst field, and f_{n-l} as the catalyst frequency.

From the derivation and final form of (2.5), which governs the spectral kinetic energy $\hat{E}_n = 1/2(\hat{\mathbf{u}}_n^H \hat{\mathbf{u}}_n)$, it is evident that spectral energy transfer to frequency f_n results directly from momentum transfer by the convective field, $\hat{\mathbf{c}}_{l \rightarrow n}$. The donor–recipient interpretation from f_l to f_n therefore directly translates to the transfer of energy, and can, in fact, be derived from the inter-triad energy conservation discussed in § 2.2.2. Specifically, under certain conditions, the energy transfer by triad $(n - l, l, n)$ is exactly equal and opposite to that by the triad $(l - n, n, l)$ and hence independent of the catalyst component. This second, rigorous motivation of the recipient–donor interpretation is elaborated on in § 2.2.3.

2.1.3. Modal decomposition

The recipient–donor interpretation of the spectral momentum and kinetic energy equations, as summarised in § 2.1.2, establishes the relationship where $\hat{\mathbf{u}}_n$ acts as the recipient of momentum or energy through the action of the convective field, $\hat{\mathbf{c}}_{l \rightarrow n}$. The objective of the decomposition is to identify the flow structures that, on average, contribute most to the momentum transfer. This objective can be framed as an optimisation problem using the calculus of variations, where a pair of jointly optimal modal bases for a given triad, (l, n) , are computed such that the convective term, $\hat{\mathbf{c}}_{l \rightarrow n}$, and the recipient, $\hat{\mathbf{u}}_n$, are

optimally represented in terms of their covariance. The kinetic energy of the convective term and the recipient are measured in the norm $\| \cdot \|$, induced by the inner product

$$\langle \mathbf{u}_1(\mathbf{x}), \mathbf{u}_2(\mathbf{x}) \rangle = \int_{\Omega} \mathbf{u}_2^H(\mathbf{x}) \mathbf{u}_1(\mathbf{x}) \, d\mathbf{x}. \tag{2.7}$$

The optimal convective–recipient covariance for each donor–recipient pair, $\sigma(l, n) \in \mathbb{R}_{\geq 0}$, is given by the optimisation problem

$$\sigma(l, n) = \max \frac{E \left\{ \langle \hat{\mathbf{c}}_{l \rightarrow n}(\mathbf{x}), \hat{\boldsymbol{\psi}}_{l \rightarrow n}(\mathbf{x}) \rangle \langle \hat{\mathbf{u}}_n(\mathbf{x}'), \hat{\boldsymbol{\phi}}_n(\mathbf{x}') \rangle^* \right\}}{\| \hat{\boldsymbol{\psi}}_{l \rightarrow n}(\mathbf{x}) \| \| \hat{\boldsymbol{\phi}}_n(\mathbf{x}') \|} \tag{2.8a}$$

$$= \max \frac{E \left\{ \left(\int_{\Omega} \hat{\boldsymbol{\psi}}_{l \rightarrow n}^H(\mathbf{x}) \hat{\mathbf{c}}_{l \rightarrow n}(\mathbf{x}) \, d\mathbf{x} \right) \left(\int_{\Omega} \hat{\mathbf{u}}_n^H(\mathbf{x}') \hat{\boldsymbol{\phi}}_n(\mathbf{x}') \, d\mathbf{x}' \right) \right\}}{\| \hat{\boldsymbol{\psi}}_{l \rightarrow n}(\mathbf{x}) \| \| \hat{\boldsymbol{\phi}}_n(\mathbf{x}') \|} \tag{2.8b}$$

$$= \max \frac{E \left\{ \iint_{\Omega} \hat{\boldsymbol{\psi}}_{l \rightarrow n}^H(\mathbf{x}) \left(\hat{\mathbf{c}}_{l \rightarrow n}(\mathbf{x}) \hat{\mathbf{u}}_n^H(\mathbf{x}') \right) \hat{\boldsymbol{\phi}}_n(\mathbf{x}') \, d\mathbf{x} \, d\mathbf{x}' \right\}}{\| \hat{\boldsymbol{\psi}}_{l \rightarrow n}(\mathbf{x}) \| \| \hat{\boldsymbol{\phi}}_n(\mathbf{x}') \|} \tag{2.8c}$$

$$= \max \frac{\iint_{\Omega} \hat{\boldsymbol{\psi}}_{l \rightarrow n}^H(\mathbf{x}) \mathbf{S}(\mathbf{x}, \mathbf{x}'; l, n) \hat{\boldsymbol{\phi}}_n(\mathbf{x}') \, d\mathbf{x} \, d\mathbf{x}'}{\| \hat{\boldsymbol{\psi}}_{l \rightarrow n}(\mathbf{x}) \| \| \hat{\boldsymbol{\phi}}_n(\mathbf{x}') \|}, \tag{2.8d}$$

where

$$\mathbf{S}(\mathbf{x}, \mathbf{x}'; l, n) = E \left\{ \hat{\mathbf{c}}_{l \rightarrow n}(\mathbf{x}) \hat{\mathbf{u}}_n^H(\mathbf{x}') \right\} \tag{2.9}$$

represents the two-point cross-bispectral covariance tensor. This tensor is a Fredholm kernel, and the modes that jointly maximise σ can be obtained via the singular value expansion (SVE) of \mathbf{S} (Schmidt 1907), that is

$$\mathbf{S}(\mathbf{x}, \mathbf{x}'; l, n) = \sum_{j=1}^{\infty} \sigma_j \hat{\boldsymbol{\psi}}_{l \rightarrow n, j}(\mathbf{x}) \hat{\boldsymbol{\phi}}_{n, j}^H(\mathbf{x}'). \tag{2.10}$$

The reality and non-negativeness of σ are guaranteed by the SVE. The convective mode, $\hat{\boldsymbol{\psi}}_{l \rightarrow n, j}$, and recipient mode, $\hat{\boldsymbol{\phi}}_{n, j}$, are ordered by their associated singular values, $\sigma_1 \geq \sigma_2 \geq \dots \geq 0$. They are also orthogonal in their respective inner products and have unit energy, that is, $\langle \hat{\boldsymbol{\psi}}_{l \rightarrow n, i}, \hat{\boldsymbol{\psi}}_{l \rightarrow n, j} \rangle = \langle \hat{\boldsymbol{\phi}}_{n, i}, \hat{\boldsymbol{\phi}}_{n, j} \rangle = \delta_{ij}$. In practice, that is for discrete data, (2.10) is solved by the singular value decomposition. In the special case where the convective term is identical to the recipient, $\hat{\mathbf{c}}_{l \rightarrow n} = \hat{\mathbf{u}}_n$, (2.8)–(2.10) reduce to the POD framework by Lumley (1967, 1970). By defining the expansion coefficients as the projections of the convective term and the recipient onto their respective modal bases (suppressing their dependence on l and n for clarity),

$$a_j = \langle \hat{\mathbf{u}}_n, \hat{\boldsymbol{\phi}}_{n, j} \rangle \quad \text{and} \quad b_j = \langle \hat{\mathbf{c}}_{l \rightarrow n}, \hat{\boldsymbol{\psi}}_{l \rightarrow n, j} \rangle, \tag{2.11}$$

we obtain the jointly optimal, ranked decomposition,

$$\hat{\mathbf{u}}_n = \sum_{j=1}^{\infty} a_j \hat{\boldsymbol{\phi}}_{n, j} \quad \text{and} \quad \hat{\mathbf{c}}_{l \rightarrow n} = \sum_{j=1}^{\infty} b_j \hat{\boldsymbol{\psi}}_{l \rightarrow n, j}. \tag{2.12}$$

The expansion coefficients, a and b , satisfy

$$E \{ b_j a_k^* \} = \sigma_j \delta_{jk}. \tag{2.13}$$

In other words, the coefficients are uncorrelated with one another. Since the coefficients are complex quantities that retain phase information, the covariance between a_j and b_j , i.e. the singular value, additionally measures the degree of phase coupling between the projections of the recipient, $\hat{\mathbf{u}}_n$, and the convective term, $\hat{\mathbf{c}}_{l \rightarrow n}$.

The expansion coefficients are also a convenient means to retrieve the spectral energy distribution of the velocity. From the mutual orthogonality of the modes, it is easily shown that the velocity power spectrum, $E\{\|\hat{\mathbf{u}}_n\|^2\}$, is recovered from the sum of the squared recipient mode coefficients

$$E\{\|\hat{\mathbf{u}}_n\|^2\} = \sum_{j=1}^{\infty} E\{a_j a_j^*\}. \tag{2.14}$$

Equation (2.14) holds for any triad (l, n) and indicates that the total power spectrum can be decomposed into contributions from individual modal ranks. The power spectra previously shown in figure 1 are obtained using this equality.

To obtain convergent estimates of the bispectral densities, we adapt Welch’s method (Welch 1967) in the decomposition framework to construct an ensemble of realisations of the temporal Fourier transform of the data from a single time series consisting of N_f snapshots based on the ergodicity hypothesis for statistically stationary flows. First, the data are segmented into N_{blk} (potentially overlapping by N_{ovlp} snapshots) blocks containing N_f consecutive snapshots separated by time step Δt . Each snapshot is represented by a column vector of length N , corresponding to the number of spatial degrees of freedom times the number of velocity components, i.e. the dimension of the data. The m th block is defined as

$$\mathbf{U}^{(m)} = \begin{bmatrix} \mathbf{u}_1^{(m)} & \mathbf{u}_2^{(m)} & \dots & \mathbf{u}_{N_f}^{(m)} \end{bmatrix} \in \mathbb{C}^{N \times N_f}, \tag{2.15}$$

and its temporal, i.e. row-wise, discrete Fourier transform (DFT) is denoted by

$$\hat{\mathbf{U}}^{(m)} = \begin{bmatrix} \hat{\mathbf{u}}_1^{(m)} & \hat{\mathbf{u}}_2^{(m)} & \dots & \hat{\mathbf{u}}_{N_f}^{(m)} \end{bmatrix}. \tag{2.16}$$

This is the same spectral estimation technique commonly used in spectral proper orthogonal decomposition (SPOD), and readers are referred to Towne, Schmidt & Colonius (2018) and Schmidt & Colonius (2020) for more details and best practices. Next, the data matrices for the recipient and convective terms

$$\hat{\mathbf{U}}_n = \begin{bmatrix} \hat{\mathbf{u}}_n^{(1)} & \hat{\mathbf{u}}_n^{(2)} & \dots & \hat{\mathbf{u}}_n^{(N_{blk})} \end{bmatrix}, \tag{2.17a}$$

$$\begin{aligned} \hat{\mathbf{C}}_{l \rightarrow n} &= \begin{bmatrix} \hat{\mathbf{c}}_{l \rightarrow n}^{(1)} & \hat{\mathbf{c}}_{l \rightarrow n}^{(2)} & \dots & \hat{\mathbf{c}}_{l \rightarrow n}^{(N_{blk})} \end{bmatrix} \\ &= - \left[\left(\hat{\mathbf{u}}_{n-l}^{(1)} \cdot \nabla \right) \hat{\mathbf{u}}_l^{(1)} \quad \left(\hat{\mathbf{u}}_{n-l}^{(2)} \cdot \nabla \right) \hat{\mathbf{u}}_l^{(2)} \quad \dots \quad \left(\hat{\mathbf{u}}_{n-l}^{(N_{blk})} \cdot \nabla \right) \hat{\mathbf{u}}_l^{(N_{blk})} \right], \end{aligned} \tag{2.17b}$$

are assembled from the columns of $\hat{\mathbf{U}}^{(m)}$. Then, the two-point cross-bispectral covariance tensor at triad (l, n) can be approximated as

$$\mathbf{S}_{l,n} = \frac{1}{N_{blk}} \hat{\mathbf{C}}_{l \rightarrow n} \hat{\mathbf{U}}_n^H. \tag{2.18}$$

Using this approximation, the infinite-dimensional SVE problem in (2.10) reduces to an $N \times N$ matrix SVD problem

$$\mathbf{W}^{1/2} \mathbf{S}_{l,n} \mathbf{W}^{1/2} = \tilde{\Psi}_{l \rightarrow n} \Sigma_{l \rightarrow n} \tilde{\Phi}_n^H, \tag{2.19}$$

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for the convective–recipient covariances, $\Sigma_{l,n} = \text{diag}([\sigma_1, \sigma_2, \dots, \sigma_{N_{blk}}])$, captured by each mode pair. The convective and recipient modes can be recovered as

$$\hat{\Psi}_{l \rightarrow n} = \mathbf{W}^{-1/2} \tilde{\Psi}_{l \rightarrow n} \quad \text{and} \quad \hat{\Phi}_n = \mathbf{W}^{-1/2} \tilde{\Phi}_n, \quad (2.20)$$

respectively, and \mathbf{W} is a weight matrix that accounts for the quadrature weights. In practice, there is no need to directly solve this SVD problem as the matrix \mathbf{S} is low-rank with a rank of $N_{blk} \ll N$. Instead, we first perform the QR decomposition of the weighted convective term data matrix

$$\frac{1}{N_{blk}} \mathbf{W}^{1/2} \hat{\mathbf{C}}_{l \rightarrow n} = \mathbf{C}_\perp \mathbf{C}_\Delta, \quad (2.21)$$

where $\mathbf{C}_\perp \in \mathbb{C}^{N \times N_{blk}}$ is orthonormal and $\mathbf{C}_\Delta \in \mathbb{C}^{N_{blk} \times N_{blk}}$ is upper triangular. We then compute the eigenvalue decomposition of the Hermitian matrix $\mathbf{C}_\Delta \hat{\mathbf{U}}_n^H \mathbf{W} \hat{\mathbf{U}}_n \mathbf{C}_\Delta^H \in \mathbb{C}^{N_{blk} \times N_{blk}}$ such that

$$\left(\mathbf{C}_\Delta \hat{\mathbf{U}}_n^H \mathbf{W} \hat{\mathbf{U}}_n \mathbf{C}_\Delta^H \right) \Psi_{l \rightarrow n} = \Psi_{l \rightarrow n} \Lambda_{l \rightarrow n}. \quad (2.22)$$

The convective modes, recipient modes and associated singular values can then be recovered as

$$\hat{\Psi}_{l \rightarrow n} = \mathbf{W}^{-1/2} \mathbf{C}_\perp \Psi_{l \rightarrow n}, \quad (2.23)$$

$$\hat{\Phi}_n = \hat{\mathbf{U}}_n \mathbf{C}_\Delta^H \Psi_{l \rightarrow n} \Lambda_{l \rightarrow n}^{-1/2}, \quad (2.24)$$

$$\Sigma_{l \rightarrow n} = \Lambda_{l \rightarrow n}^{1/2}, \quad (2.25)$$

respectively (the implicit dependence of $\hat{\Phi}_n$ on l is suppressed for readability). Alternatively, they form the weighted SVD of the sample cross-bispectral covariance matrix

$$\mathbf{S}_{l,n} = \hat{\Psi}_{l \rightarrow n} \Sigma_{l \rightarrow n} \hat{\Phi}_n^H \quad \text{with} \quad \hat{\Phi}_n^H \mathbf{W} \hat{\Phi}_n = \hat{\Psi}_{l \rightarrow n}^H \mathbf{W} \hat{\Psi}_{l \rightarrow n} = \mathbf{I}. \quad (2.26)$$

The expansion coefficients associated with the recipient and convective modes are respectively obtained by orthogonal projection

$$\mathbf{A}_{l,n} = \hat{\Phi}_n^H \mathbf{W} \hat{\mathbf{U}}_n \quad \text{and} \quad \mathbf{B}_{l,n} = \hat{\Psi}_{l \rightarrow n}^H \mathbf{W} \hat{\mathbf{C}}_{l \rightarrow n}, \quad (2.27)$$

from which the data matrices may be reconstructed as

$$\hat{\mathbf{U}}_n = \hat{\Phi}_n \mathbf{A}_{l,n} \quad \text{and} \quad \hat{\mathbf{C}}_{l \rightarrow n} = \hat{\Psi}_{l \rightarrow n} \mathbf{B}_{l,n}. \quad (2.28)$$

2.1.4. Donor and catalyst modes as an aid to interpretation

Within the momentum equations (2.3), the catalyst and donor spectral components, $\hat{\mathbf{u}}_{n-l}$ and $\hat{\mathbf{u}}_l$, are subsumed under the convective term, $\hat{\mathbf{c}}_{l \rightarrow n}$, and represented collectively by a linear combination of convective modes, $\hat{\Psi}_{l \rightarrow n}$. The latter satisfy the properties of optimality (in terms of covariance) and orthonormality as given in § 2.1.3. As a pure post-processing step, and only for the purpose of physical interpretation, it may also be instructive to visualise the coherent structures that occupy the subspaces spanned separately by $\hat{\mathbf{u}}_{n-l}$ and $\hat{\mathbf{u}}_l$. We call these structures catalyst modes, $\hat{\xi}_{n-l}$, and donor modes, $\hat{\xi}_l$, respectively. Although neither $\hat{\xi}_{n-l}$ nor $\hat{\xi}_l$ forms an orthogonal basis, they are linked rigorously to the convective and recipient modes via the expansion coefficients, a and b , in a manner reminiscent of extended POD (Borée 2003) and extended SPOD (Towne *et al.* 2015; Karban *et al.* 2022). From the expansion (2.12) and the uncorrelatedness property

(2.13), we can rewrite the convective mode as

$$\hat{\psi}_{l \rightarrow n, j} = \frac{E\{\hat{c}_{l \rightarrow n} a_j^*\}}{\sigma_j}. \quad (2.29)$$

Following the form of this identity, we define the donor mode

$$\hat{\xi}_{l, j} \equiv \frac{E\{\hat{u}_l a_j^*\}}{\sigma_j}. \quad (2.30)$$

From (2.12), the portion of the recipient field associated with the j th recipient mode is

$$\hat{u}_{n, j} = a_j \hat{\phi}_{n, j}, \quad (2.31)$$

and similarly for the convective field and j th convective mode

$$\hat{c}_{l \rightarrow n, j} = b_j \hat{\psi}_{l \rightarrow n, j}. \quad (2.32)$$

On the pattern of (2.32), we also define the portion of the donor field associated with the j th donor mode

$$\hat{u}_{l, j} \equiv b_j \hat{\xi}_{l, j}. \quad (2.33)$$

A key property is that, for each triad, the donor and convective modes share the same expansion coefficient, b_j . We can now establish that the donor mode, $\hat{\xi}_{l, j}$, represents the only part of the donor field that is correlated with the partial recipient field, $\hat{u}_{n, j}$. The correlation of the full donor field with the partial recipient field is

$$E\{\hat{u}_l \hat{u}_{n, j}^H\} = E\{\hat{u}_l a_j^* \hat{\phi}_{n, j}^H\} = E\{\underbrace{\hat{u}_l a_j^*}_{\text{using (2.30)}} \hat{\phi}_{n, j}^H\} = \sigma_j \hat{\xi}_{l, j} \hat{\phi}_{n, j}^H. \quad (2.34)$$

The correlation of the partial donor field with the partial recipient field is

$$E\{\hat{u}_{l, j} \hat{u}_{n, j}^H\} = E\{b_j \hat{\xi}_{l, j} a_j^* \hat{\phi}_{n, j}^H\} = E\{b_j a_j^* \hat{\xi}_{l, j} \hat{\phi}_{n, j}^H\} = \sigma_j \hat{\xi}_{l, j} \hat{\phi}_{n, j}^H = E\{\hat{u}_l \hat{u}_{n, j}^H\}, \quad (2.35)$$

which completes the proof. This property of exclusive correlation with the recipient applies also to the catalyst mode, which we define as

$$\hat{\xi}_{n-l, j} \equiv \frac{E\{\hat{u}_{n-l} a_j^*\}}{\sigma_j}. \quad (2.36)$$

In matrix form, the donor and catalyst modes defined in (2.30) and (2.36) are respectively given by

$$\hat{\Xi}_l \equiv \frac{1}{N_{blk}} \hat{\mathbf{U}}_l \mathbf{A}_{l, n}^H \boldsymbol{\Sigma}_{l \rightarrow n}^{-1} \quad \text{and} \quad \hat{\Xi}_{n-l} \equiv \frac{1}{N_{blk}} \hat{\mathbf{U}}_{n-l} \mathbf{A}_{l, n}^H \boldsymbol{\Sigma}_{l \rightarrow n}^{-1}. \quad (2.37)$$

We emphasise that the donor and catalyst modes, $\hat{\xi}_{l, j}$ and $\hat{\xi}_{n-l, j}$, do not form part of the optimisation problem (2.8), and in general cannot be used to reconstruct the convective mode, $\hat{\psi}_{l \rightarrow n, j}$. Rather, the donor and catalyst modes derive significance only from their statistical correlation with the recipient field, as given by (2.34) and (2.35). In principle, it is possible to define $\hat{\xi}_{l, j}$ and $\hat{\xi}_{n-l, j}$ in terms of b_j^* instead, yielding donor and catalyst modes that are correlated with the convective field. However, we are interested in flow structures that are correlated with the physically observable velocity field, i.e. the recipient field. A similar choice is made by Karban *et al.* (2022) in the context of SPOD; expansion

coefficients obtained from an optimal decomposition of the observable are utilised to find forcing structures correlated with the observable. By defining donor and catalyst modes, we can now visualise triadic mode interactions in the form of physically interpretable Feynman-type diagrams.

2.1.5. Relationship to other methods

In the literature, several data-driven methods bear similarities to TOD. Here, we briefly outline these methods without delving into extensive detail. The optimisation problem in (2.8) can be interpreted as maximising the covariance between the expansion coefficients, expressed as $\sigma_j = \max E\{b_j a_j^*\}$, suggesting that TOD can be seen as a specialised form of maximum covariance analysis (MCA) (von Storch & Zwiers 1999). The MCA, along with the closely related canonical correlation analysis (CCA) (Hotelling 1992), identifies bases that are jointly optimal in multivariate statistical terms and is widely applied in data mining. For further details on CCA and its variations, see Thompson (1984), Hardoon, Szedmak & Shawe-Taylor (2004) and Andrew *et al.* (2013).

The TOD and BMD (Schmidt 2020) methods can be compared as both are bispectral methods that directly consider third-order statistics. As discussed in the introduction (§ 1), BMD maximises the spatial integral of the classical, one-dimensional bispectrum, whereas TOD explicitly distinguishes between convective and recipient terms and maximises their covariance. Despite these differences, both methods yield qualitatively consistent results. A related approach is cross-POD (CPOD; Cavalieri & da Silva 2021), which, despite its name, maximises a spatial integral rather than a correlation and therefore aligns more closely with BMD. Mathematically, BMD addresses the numerical radius problem, while CPOD solves for the numerical abscissa. The CPOD method could also be adapted to the frequency domain for specialised triad identification.

The variational formulation of TOD in (2.8) suggests connections to classical POD and its various extensions. Balanced POD (Rowley 2005) seeks a pair of bi-orthogonal modal bases that optimally represent the controllability and observability Gramians in a linear input–output system. The TOD method can also be related to frequency-domain POD variants: SPOD (Towne *et al.* 2018; Schmidt & Colonius 2020) and cyclostationary SPOD (CS-SPOD; Heidt & Colonius 2024b). While SPOD optimally captures power but assumes frequency independence, it has been recently applied to study modal energy transfer (Nekkanti, Colonius & Schmidt 2025). The CS-SPOD method extends SPOD by including correlations between frequency components that become triadically linked in the presence of a time-periodic mean flow. An additional, but partially operator-driven extension for analysing triadic interactions is the resolvent-based extended SPOD (Towne *et al.* 2015; Karban *et al.* 2022, 2024), which finds modes representing the convective term that share the same expansion coefficients as the usual SPOD modes.

2.2. Modal energy flow analysis

The nonlinear term in the Navier–Stokes equations is conservative (see e.g. Schmid & Henningson 2001). Unlike the linear terms, which may inject energy into or siphon energy out of the system, the nonlinear term only redistributes energy. Collectively, the net effect of all nonlinear triadic interactions is thus to conserve energy. Energy is also conserved by individual groups of triads (Kraichnan 1959). Based on pairwise triadic conservation, Smyth (1992) proposed a methodology by which the energy exchanged between two arbitrary frequencies can be systematically quantified. We demonstrate the utility of this method for TOD and identify and quantify energy propagation via mode interactions.

2.2.1. Spectral energy budget

Taking the real part of (2.5) yields the conservation equation for the spectral kinetic energy

$$0 = \mathcal{R} \left\{ \underbrace{-\hat{\mathbf{u}}_n^H \sum_{l=-\infty}^{\infty} (\hat{\mathbf{u}}_{n-l} \cdot \nabla) \hat{\mathbf{u}}_l}_{\hat{T}_n} \right\} + \mathcal{R} \left\{ \underbrace{-\nabla \cdot (\hat{p}_n \hat{\mathbf{u}}_n^H)}_{\hat{F}_n} \right\} + \mathcal{R} \left\{ \underbrace{\frac{1}{Re} \hat{\mathbf{u}}_n^H \nabla^2 \hat{\mathbf{u}}_n}_{\hat{D}_n} \right\}, \quad (2.38)$$

which partitions the budget of the kinetic energy at frequency f_n , $\hat{E}_n = 1/2(\hat{\mathbf{u}}_n^H \hat{\mathbf{u}}_n)$, into contributions from inter-scale transfer, \hat{T}_n , pressure work, \hat{F}_n , and dissipation, \hat{D}_n . The transfer term may be expressed as the summation over individual triads

$$\hat{T}_n = \sum_{l=-\infty}^{\infty} \hat{T}_{l \rightarrow n}, \quad (2.39)$$

where $\hat{T}_{l \rightarrow n} \equiv -\hat{\mathbf{u}}_n^H (\hat{\mathbf{u}}_{n-l} \cdot \nabla) \hat{\mathbf{u}}_l$. The inter-scale transfer can be further partitioned into advection, \hat{A}_n , production, \hat{P}_n , and nonlinear transfer, $\hat{T}_{NL,n}$

$$\hat{T}_n = \hat{T}_{n \rightarrow n} + \hat{T}_{0 \rightarrow n} + \sum_{l \neq 0, n} \hat{T}_{l \rightarrow n} \quad (2.40a)$$

$$= \underbrace{-\hat{\mathbf{u}}_n^H (\bar{\mathbf{u}} \cdot \nabla) \hat{\mathbf{u}}_n}_{\hat{A}_n} - \underbrace{\hat{\mathbf{u}}_n^H (\hat{\mathbf{u}}_n \cdot \nabla) \bar{\mathbf{u}}}_{\hat{P}_n} - \underbrace{\hat{\mathbf{u}}_n^H \sum_{l \neq 0, n} (\hat{\mathbf{u}}_{n-l} \cdot \nabla) \hat{\mathbf{u}}_l}_{\hat{T}_{NL,n}}. \quad (2.40b)$$

The terms \hat{A}_n and \hat{P}_n are triads in which one component is the mean flow, $\bar{\mathbf{u}}$. They are, therefore, linear mechanisms with respect to the mean. The budget of \hat{T}_n is summarised graphically in the bispectrum in figure 3(a), in which \hat{A}_n and \hat{P}_n are naturally recovered as special cases of $\hat{T}_{l \rightarrow n}$. Along $l = n$, $\hat{T}_{l \rightarrow n} = \hat{A}_n$ and is indicated in magenta. Along $l = 0$, $\hat{T}_{l \rightarrow n} = \hat{P}_n$ and is indicated in green. In the remainder of the bispectrum, quadratic nonlinearities are active.

For the special case of $n = 0$, (2.38) simplifies to

$$0 = -\bar{\mathbf{u}}^T \left((\bar{\mathbf{u}} \cdot \nabla) \bar{\mathbf{u}} + \sum_{l \neq 0} \mathcal{R} \left\{ \underbrace{(\hat{\mathbf{u}}_{-l} \cdot \nabla) \hat{\mathbf{u}}_l}_{-\hat{c}_{l \rightarrow 0}} \right\} \right) - \nabla \cdot (\bar{p} \bar{\mathbf{u}}) + \frac{1}{Re} \bar{\mathbf{u}}^T \nabla^2 \bar{\mathbf{u}} \quad (2.41a)$$

$$= \underbrace{-(\bar{\mathbf{u}} \cdot \nabla) \frac{1}{2} \bar{\mathbf{u}}^T \bar{\mathbf{u}}}_{\bar{A}} - \underbrace{\nabla \cdot \overline{(\mathbf{u}'(\mathbf{u}')^T \bar{\mathbf{u}})}}_{\bar{T}_{NL}} + \underbrace{\overline{\mathbf{u}'(\mathbf{u}')^T} : \nabla \bar{\mathbf{u}}}_{-\bar{P}} - \underbrace{\nabla \cdot (\bar{p} \bar{\mathbf{u}})}_{\bar{F}} + \underbrace{\frac{1}{Re} \bar{\mathbf{u}}^T \nabla^2 \bar{\mathbf{u}}}_{\bar{D}}, \quad (2.41b)$$

which is the evolution equation for the MKE, $\bar{E} = 1/2(\bar{\mathbf{u}}^T \bar{\mathbf{u}})$. The terms labelled \bar{A} , \bar{T}_{NL} , \bar{P} , \bar{F} and \bar{D} are the contributions to the MKE from advection, transfer, production, pressure work and dissipation, respectively. The mean production can alternatively be expressed as a summation of the spectral production

$$\bar{P} = \sum_{n \neq 0} \hat{P}_n = - \sum_{n \neq 0} \hat{\mathbf{u}}_n^H (\hat{\mathbf{u}}_n \cdot \nabla) \bar{\mathbf{u}} = -\overline{\mathbf{u}'(\mathbf{u}')^T} : \nabla \bar{\mathbf{u}}, \quad (2.42)$$

where we have invoked the general form of Parseval's theorem to arrive at the last step. The MKE (2.41) is the counterpart to the RANS (2.4). The MKE budget can similarly

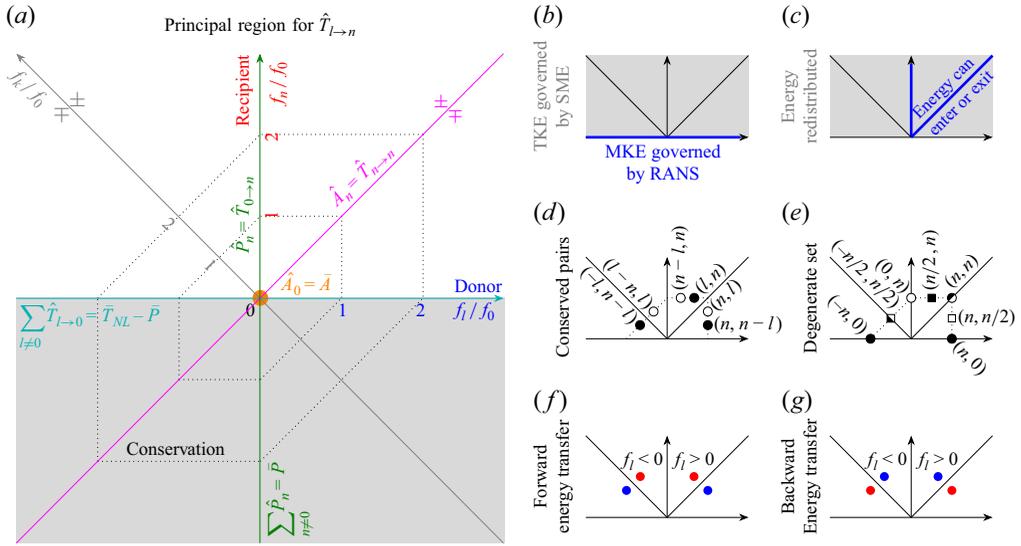


Figure 3. Interpretation guide for inter-scale energy transfer. (a) Schematic of triadic energy transfer, $\hat{T}_{l \rightarrow n}$, in the bispectral plane with the donor frequency, f_l , as abscissa (blue) and the recipient frequency, f_n , as ordinate (red), both normalised by f_0 , the fundamental frequency. Only the principal region has to be considered; the grey bottom half-plane contains redundant information. The catalyst frequency, $f_k = f_{n-l}$, is indicated as a grey line. $\hat{T}_{l \rightarrow n}$ is conserved on nested hexagons with dotted lines exemplifying this conservation for $f_l = f_n = f_0$ and $f_l = f_n = 2f_0$. Further indicated are spectral TKE and MKE contributions from linear advection by the mean (magenta), production (green), mean self-advection (orange) and transfer-production difference (teal). The magenta \pm symbols denote the property of pairwise conservation about $f_l = f_n$; similarly, the grey \pm symbols denote conservation about $f_l = -f_n$. Panels (b–g) are interpretation aids.

be inferred from figure 3(a). Specifically, the mean transfer and production, \bar{T}_{NL} and \bar{P} , respectively, can be recovered from $\hat{T}_{l \rightarrow n}$ along the $f_n = 0$ and $f_l = 0$ axes. The mean advection, $\bar{A} = \hat{T}_{0 \rightarrow 0}$, is located at the origin. Together, (2.4) and (2.41) describe the deformation of the mean flow in terms of its momentum and energy, respectively. In particular, the convective term, $\hat{c}_{l \rightarrow 0}$, and its corresponding convective mode, $\hat{\psi}_{l \rightarrow 0}$, for all l conspire to alter the mean flow by removing momentum and energy from the mean.

To aid in the interpretation of the bispectrum, each panel in figure 3(b–g) highlights a distinct concept. Figure 3(b) illustrates the region of the bispectrum plane governed by the spectral momentum equations (SME), $f_n \neq 0$, and the region governed by the RANS equations, $f_n = 0$. Figure 3(c) illustrates the region where energy is permitted to enter or exit the budget, and the region where energy is merely redistributed. Figure 3(d) illustrates prototypical conserved triad pairs, with each pair denoted by a filled and open circle. The pairs are: (l, n) and (n, l) , $(n-l, n)$ and $(n, n-l)$, and $(-l, n-l)$ and $(l-n, l)$. Together they form a six-triad conserved set. Depending on the frequencies f_l and f_n , neighbouring triads in figure 3(d) may merge into the degenerate sets in figure 3(e). Specifically, for $f_l = f_n$, the six-triad set degenerates into a four-triad conserved set, (n, n) , $(0, n)$, $(n, 0)$ and $(-n, 0)$, marked by circles; for $f_l = f_n/2$, the six-triad set instead degenerates into a three-triad conserved set, $(n/2, n)$, $(n, n/2)$ and $(-n/2, n/2)$, marked by squares. Under certain conditions, the triads (n, n) and $(-n/2, n/2)$, marked by half-filled symbols, each acts as a conserved repeated pair. Finally, figure 3(f) illustrates an arrangement of triads with positive (red) and negative (blue) energy transfers that indicates forward energy transfer from low to high frequencies, or large to small scales, while

figure 3(g) indicates backward transfer from high to low frequencies, or small to large scales. We will expand on these concepts in the following sections.

2.2.2. Inter-triad conservation of spectral energy

While the nonlinear term in the momentum equations can transfer and redistribute kinetic energy, it can neither produce nor remove net energy. The conservation of scale-to-scale kinetic energy transfer outlined in the following has been known at least as far back as the early text of Batchelor (1953). Here, we closely follow the derivation presented by Barthel (2022). It is common practice to assume that boundary contributions to $\hat{\mathcal{T}}_{l \rightarrow n}$ vanish, typically due to steady, no-slip or periodic boundary conditions. However, for most numerical or experimental flow data where fluid enters and exits through the domain boundaries, these conditions do not hold. Therefore, we include the boundary term in our derivation. The significance of boundary contributions will become clear in the applications discussed in § 3.

We define the integral energy transfer

$$\hat{\mathcal{T}}_{l \rightarrow n} \equiv \int_{\Omega} \hat{\mathcal{T}}_{l \rightarrow n} dx = - \int_{\Omega} \hat{\mathbf{u}}_n^H (\hat{\mathbf{u}}_{n-l} \cdot \nabla) \hat{\mathbf{u}}_l dx \quad (2.43)$$

to quantify the energy transferred by the triad $(n-l, l, n)$. Integration by parts leads to

$$\hat{\mathcal{T}}_{l \rightarrow n} = - \underbrace{\int_{\Omega} \nabla \cdot \left[\left(\hat{\mathbf{u}}_n^H \hat{\mathbf{u}}_l \right) \hat{\mathbf{u}}_{n-l} \right] dx}_{F_{l \rightarrow n}} + \underbrace{\int_{\Omega} \hat{\mathbf{u}}_l^T \nabla \cdot \left(\hat{\mathbf{u}}_n^* \hat{\mathbf{u}}_{n-l}^T \right) dx}_{R_2}. \quad (2.44)$$

The term $F_{l \rightarrow n}$ can be expressed using the divergence theorem as

$$F_{l \rightarrow n} = - \oint_{\partial \Omega} \left(\hat{\mathbf{u}}_n^H \hat{\mathbf{u}}_l \right) \hat{\mathbf{u}}_{n-l} \cdot \mathbf{n} dl, \quad (2.45)$$

which can be interpreted as the real part of the total bispectral density flux into the control volume, Ω . The term R_2 can be expanded using the product rule as

$$R_2 = \underbrace{\int_{\Omega} \hat{\mathbf{u}}_l^T \hat{\mathbf{u}}_n^* \nabla \cdot \hat{\mathbf{u}}_{n-l} dx}_{R_3} + \underbrace{\int_{\Omega} \hat{\mathbf{u}}_l^T (\hat{\mathbf{u}}_{n-l} \cdot \nabla) \hat{\mathbf{u}}_n^* dx}_{R_4}. \quad (2.46)$$

The term R_3 is zero due to incompressibility, $\nabla \cdot \hat{\mathbf{u}} = 0$. For real data, for which $\hat{\mathbf{u}}_{l-n} = \hat{\mathbf{u}}_{n-l}^*$ by the conjugate symmetry of the Fourier transform, and using the equation

$$\hat{\mathcal{T}}_{n \rightarrow l} = - \int_{\Omega} \hat{\mathbf{u}}_l^H (\hat{\mathbf{u}}_{l-n} \cdot \nabla) \hat{\mathbf{u}}_n dx = - \int_{\Omega} \hat{\mathbf{u}}_l^H (\hat{\mathbf{u}}_{n-l}^* \cdot \nabla) \hat{\mathbf{u}}_n dx = -R_4^*, \quad (2.47)$$

we can obtain the pairwise relationships

$$\hat{\mathcal{T}}_{l \rightarrow n} + \hat{\mathcal{T}}_{n \rightarrow l}^* = F_{l \rightarrow n} = F_{n \rightarrow l} \quad (2.48a)$$

$$\text{and } \hat{\mathcal{T}}_{l \rightarrow n}^{\mathcal{R}} + \hat{\mathcal{T}}_{n \rightarrow l}^{\mathcal{R}} = F_{l \rightarrow n}^{\mathcal{R}} = F_{n \rightarrow l}^{\mathcal{R}}, \quad (2.48b)$$

where we defined the shorthand $(\cdot)^{\mathcal{R}} \equiv \mathcal{R}\{\cdot\}$. If the flux $F_{l \rightarrow n} = F_{n \rightarrow l}$ vanishes over the boundary $\partial \Omega$, the net real energy transfer is pairwise-conserved, $\hat{\mathcal{T}}_{l \rightarrow n}^{\mathcal{R}} + \hat{\mathcal{T}}_{n \rightarrow l}^{\mathcal{R}} = 0$. Three such conserved pairs then form the six-triad conserved set

$$\left(\hat{\mathcal{T}}_{l \rightarrow n}^{\mathcal{R}} + \hat{\mathcal{T}}_{n \rightarrow l}^{\mathcal{R}} \right) + \left(\hat{\mathcal{T}}_{n-l \rightarrow n}^{\mathcal{R}} + \hat{\mathcal{T}}_{n \rightarrow n-l}^{\mathcal{R}} \right) + \left(\hat{\mathcal{T}}_{-l \rightarrow n-l}^{\mathcal{R}} + \hat{\mathcal{T}}_{l-n \rightarrow l}^{\mathcal{R}} \right) = 0, \quad (2.49)$$

involving all permutations of the triplet $(n-l, l, n)$.

From the pairwise conservation expressed in (2.48), we can deduce that two types of triads form degenerate pairs and will have no integral energy transfer if the net flux is zero. The first type satisfies the component-wise equality $(n - l, l, n) = (l - n, n, l)$, which is equivalent to $l = n$ and corresponds to the magenta diagonal line in figure 3(a), which indicates spectral advection. Triads on either side of $l = n$ conserve energy in a pairwise fashion. The second type satisfies $(n - l, l, n) = -(l - n, n, l)$, equivalent to $l + n = 0$, corresponding to the grey line, which is also the f_k -axis. Triad pairs on either side of $l + n = 0$ are similarly conserved. In practice, the degenerate triads along the spectral advection line, $l = n$, may exhibit finite integral energy transfer, as the mean component of open flows tends to violate the zero-flux condition.

It follows from the pairwise and six-triad conservation properties in (2.48) and (2.49) that the sum of all nonlinear transfer terms is zero, i.e.

$$\sum_{n=-\infty}^{\infty} \sum_{l \neq 0, n} \hat{T}_{l \rightarrow n}^{\mathcal{R}} = 0, \tag{2.50}$$

provided that the fluxes vanish over the domain boundary as before. Nonlinear energy transfer is thus both locally conserved by a limited set of triads and globally conserved by all triads. A visual interpretation is that the whole bispectral plane is a union of all nested hexagons. This reflects the well-known conservative property of the convective term. In summary, the integral energy transfer satisfies a hierarchy of conservative properties of increasing granularity: from global, to six-triad, to pairwise conservation. Energy is permitted to enter or leave the system only via mean production and linear advection, i.e. the green and magenta lines, respectively, in figure 3(a) (as well as through dissipation and other neglected terms), and is then distributed or scattered over the remainder of the bispectrum plane.

2.2.3. Modal energy budget and directivity

Provided the flux, $F_{l \rightarrow n}$, is negligible, (2.48) is an expression of the conservative nature of spectral energy transfer (Kraichnan 1959). The transfer of energy achieved by the triad $(n - l, l, n)$ is precisely balanced by the triad $(l - n, n, l)$. This observation motivates the recipient–donor interpretation, see § 2.1.2, of the energy transfer term, $\hat{T}_{l \rightarrow n}$, in which the frequency components f_l and f_n are respectively identified as the donor and recipient of energy, and f_{n-l} as the catalyst that neither donates nor receives energy, but merely mediates the transfer. This interpretation reframes the pairwise conservation in (2.48) as the statement that the energy transferred from f_l to f_n must be equal and opposite to the energy transferred from f_n to f_l : a physically intuitive result. For any arbitrary donor–recipient pair, f_l and f_n , the direction and quantity of the spectral transfer are given by the sign and magnitude, respectively, of $\hat{T}_{l \rightarrow n}$ or $\hat{T}_{n \rightarrow l}$. This allows the inter-scale propagation of energy across all frequencies to be systematically deduced from the convective modes and recipient modes of the decomposition. The notation $l \rightarrow n$ reflects this understanding. The donor and recipient frequencies are labelled in figure 3(a) in blue and red, respectively. Positive energy transfers in the region $f_n > |f_l|$ or, equivalently, negative energy transfers in the region $f_n < |f_l|$ signal the transport of energy from low to high frequencies, that is, a forward cascade. Conversely, an inverse cascade of energy towards lower frequencies can be readily identified by negative energy transfers in the region $f_n > |f_l|$ or, equivalently, positive energy transfers in the region $f_n < |f_l|$.

Using the bilinearity of the inner product, we can expand the expected total energy transfer, abbreviated as $\hat{T}_{l \rightarrow n}^{avg} \equiv E\{\hat{T}_{l \rightarrow n}\}$ in the following, in terms of the convective

modes and recipient modes

$$\begin{aligned}
 \hat{\mathcal{T}}_{l \rightarrow n}^{avg} &= E \left\{ \hat{\mathcal{T}}_{l \rightarrow n} \right\} = E \left\{ \langle \hat{\mathbf{c}}_{l \rightarrow n}, \hat{\mathbf{u}}_n \rangle \right\} \\
 &= E \left\{ \left\langle \sum_{j=1}^{\infty} b_j \hat{\boldsymbol{\psi}}_{l \rightarrow n, j}, \sum_{j=1}^{\infty} a_j \hat{\boldsymbol{\phi}}_{n, j} \right\rangle \right\} \\
 &= E \left\{ \underbrace{\langle b_1 \hat{\boldsymbol{\psi}}_{l \rightarrow n, 1}, a_1 \hat{\boldsymbol{\phi}}_{n, 1} \rangle}_{\hat{\mathcal{T}}_{l \rightarrow n, 1 \rightarrow 1}} \right\} + E \left\{ \underbrace{\langle b_2 \hat{\boldsymbol{\psi}}_{l \rightarrow n, 2}, a_1 \hat{\boldsymbol{\phi}}_{n, 1} \rangle}_{\hat{\mathcal{T}}_{l \rightarrow n, 2 \rightarrow 1}} \right\} + E \left\{ \underbrace{\langle b_1 \hat{\boldsymbol{\psi}}_{l \rightarrow n, 1}, a_2 \hat{\boldsymbol{\phi}}_{n, 2} \rangle}_{\hat{\mathcal{T}}_{l \rightarrow n, 1 \rightarrow 2}} \right\} \\
 &\quad + E \left\{ \underbrace{\langle b_2 \hat{\boldsymbol{\psi}}_{l \rightarrow n, 2}, a_2 \hat{\boldsymbol{\phi}}_{n, 2} \rangle}_{\hat{\mathcal{T}}_{l \rightarrow n, 2 \rightarrow 2}} \right\} + \dots \\
 &= E \{ b_1 a_1^* \} \langle \hat{\boldsymbol{\psi}}_{l \rightarrow n, 1}, \hat{\boldsymbol{\phi}}_{n, 1} \rangle + E \{ b_2 a_1^* \} \langle \hat{\boldsymbol{\psi}}_{l \rightarrow n, 2}, \hat{\boldsymbol{\phi}}_{n, 1} \rangle + E \{ b_1 a_2^* \} \langle \hat{\boldsymbol{\psi}}_{l \rightarrow n, 1}, \hat{\boldsymbol{\phi}}_{n, 2} \rangle \\
 &\quad + E \{ b_2 a_2^* \} \langle \hat{\boldsymbol{\psi}}_{l \rightarrow n, 2}, \hat{\boldsymbol{\phi}}_{n, 2} \rangle + \dots \\
 &= \sigma_1 \langle \hat{\boldsymbol{\psi}}_{l \rightarrow n, 1}, \hat{\boldsymbol{\phi}}_{n, 1} \rangle + \sigma_2 \langle \hat{\boldsymbol{\psi}}_{l \rightarrow n, 2}, \hat{\boldsymbol{\phi}}_{n, 2} \rangle + \dots \\
 &= \sum_{j=1}^{\infty} \sigma_j \langle \hat{\boldsymbol{\psi}}_{l \rightarrow n, j}, \hat{\boldsymbol{\phi}}_{n, j} \rangle = \sum_{j=1}^{\infty} \hat{\mathcal{T}}_{l \rightarrow n, j}^{avg}, \tag{2.51}
 \end{aligned}$$

where we have made use of the property that the expansion coefficients are uncorrelated, see (2.13). Each term in the expansion, $\hat{\mathcal{T}}_{l \rightarrow n, j}^{avg} = \sigma_j \langle \hat{\boldsymbol{\psi}}_{l \rightarrow n, j}, \hat{\boldsymbol{\phi}}_{n, j} \rangle$, represents the average integral modal energy transfer from the j th convective mode to the j th recipient mode at frequency f_n . It follows from the uncorrelatedness of the expansion coefficients that, on average, no energy is exchanged between the i th and j th modes for $i \neq j$. For brevity, we therefore only use the subscript j to represent these transfers. An obvious consequence of the expansion (2.51) is that summing the modal transfers, $\hat{\mathcal{T}}_{l \rightarrow n, j}^{avg}$, over all j identically recovers the total transfer, $\hat{\mathcal{T}}_{l \rightarrow n}^{avg}$. The latter is simply the result of classical Fourier decomposition-based energy transfer analysis, such as the work of Freeman *et al.* (2024). While the total transfer can be informative (particularly for flows which exhibit a low-rank dynamics), it masks variations in energy transport with the rank, j , that the modal transfer captures. This distinction becomes most salient for turbulent flows, concrete examples of which will be presented in §§ 3.2–3.3.

In general, the conservation of energy described in § 2.2.2 is satisfied in aggregate for $\hat{\mathcal{T}}_{l \rightarrow n}^{avg}$, not for an individual modal transfer, $\hat{\mathcal{T}}_{l \rightarrow n, j}^{avg}$. However, $\hat{\mathcal{T}}_{l \rightarrow n}^{avg}$ will be well represented by $\hat{\mathcal{T}}_{l \rightarrow n, 1}^{avg}$ if the singular values, σ_j , decay rapidly, that is, if the leading mode pair captures the majority of the total convective–recipient covariance. In such a case, $\hat{\mathcal{T}}_{l \rightarrow n, 1}^{avg}$ should approximately be conserved, as we will demonstrate in § 3.

We can further relate the magnitude of the expected integral modal transfer, $\hat{\mathcal{T}}_{l \rightarrow n, j}^{avg}$, and that of its real part, $\mathcal{R}\{\hat{\mathcal{T}}_{l \rightarrow n, j}^{avg}\} \equiv \hat{\mathcal{T}}_{l \rightarrow n, j}^{avg, \mathcal{R}}$, to the mode bispectrum, σ_j , by writing

$$\left| \hat{\mathcal{T}}_{l \rightarrow n, j}^{avg, \mathcal{R}} \right| = \left| \mathcal{R} \left\{ \sigma_j \langle \hat{\boldsymbol{\psi}}_{l \rightarrow n, j}, \hat{\boldsymbol{\phi}}_{n, j} \rangle \right\} \right| = \sigma_j \underbrace{\left| \mathcal{R} \left\{ \langle \hat{\boldsymbol{\psi}}_{l \rightarrow n, j}, \hat{\boldsymbol{\phi}}_{n, j} \rangle \right\} \right|}_{\in [0, 1]}, \tag{2.52}$$

where we have invoked the Cauchy–Schwarz inequality in the last step to obtain the bounds on the inner product of the two (normalised) modes. In other words, the integral modal transfer is proportional to and bounded by the singular value, with the scaling factor given by the degree of spatial alignment between the convective and recipient modes. The upper bound is a physically intuitive property; a weak triadic interaction, which is associated with a small singular value, must transfer energy sparingly.

From expanding the expected integral modal transfer in terms of the recipient and convective modes according to (2.51)

$$\hat{\mathcal{T}}_{l \rightarrow n, j}^{avg} = \int_{\Omega} \sigma_j \hat{\phi}_{n, j}^H \hat{\psi}_{l \rightarrow n, j} d\mathbf{x} = \int_{\Omega} E \left\{ \hat{\mathcal{T}}_{l \rightarrow n, j} \right\} d\mathbf{x}, \quad (2.53)$$

it becomes evident that its value is determined by integrating the mode product $\hat{\phi}_{n, j}^H \hat{\psi}_{l \rightarrow n, j}$, weighted by σ_j , over the domain of interest, Ω . This motivates the definition of the integrand of $\hat{\mathcal{T}}_{l \rightarrow n, j}^{avg}$

$$\hat{\tau}_{l \rightarrow n, j} \equiv \sigma_j \hat{\phi}_{n, j}^H \hat{\psi}_{l \rightarrow n, j}, \quad (2.54)$$

as the modal transfer field (or transfer density). We note that $\hat{\tau}_{l \rightarrow n, j}$ is a complex quantity. In the discrete case, for the example of two-dimensional data, the transfer field is computed as

$$\hat{\tau}_{l \rightarrow n, j} = \sigma_j \left[\left(\hat{\phi}_{n, j}^{(u)} \right)^* \circ \hat{\psi}_{l \rightarrow n, j}^{(u)} + \left(\hat{\phi}_{n, j}^{(v)} \right)^* \circ \hat{\psi}_{l \rightarrow n, j}^{(v)} \right], \quad (2.55)$$

where the superscripts $(\cdot)^{(u)}$ and $(\cdot)^{(v)}$ respectively denote the u and v components of each mode, and \circ denotes the Hadamard product. The expression extends trivially to the three-dimensional case.

Owing to the conservation properties described in § 2.2.2, the expected integral modal transfer, $\hat{\mathcal{T}}_{l \rightarrow n, j}^{avg}$, has a principal region that spans the sector $f_n \geq |f_l|$, between the grey and magenta lines in figure 3(a). All other integral transfers can be recovered from the principal region via pairwise conservation. On the other hand, the modes, $\hat{\psi}_{l \rightarrow n, j}$ and $\hat{\phi}_{n, j}$, and transfer field, $\hat{\tau}_{l \rightarrow n, j}$, do not obey conservation, as will become clear in the examples (§ 3). Their principal region is thus the entire top half-plane in figure 3(a).

3. Results

We demonstrate the decomposition and modal energy flow analysis using three different flow cases. The first case involves numerical simulation data of the wake behind a cylinder at $Re = 100$ (Chu & Schmidt 2023), a canonical example of unsteady laminar flow with well-established nonlinear dynamics. The second case uses particle image velocimetry (PIV) data from a wind turbine wake (Biswas & Buxton 2024a,b), illustrating the application of TOD to turbulent flows, even in the presence of measurement noise. The third case considers a widely used numerical simulation database of forced isotropic turbulence in a three-dimensional periodic box (Perlman *et al.* 2007; Li *et al.* 2008). Instantaneous snapshots of the three flows are shown earlier in figure 1, while the corresponding data sizes, flow parameters and spectral estimation settings are summarised in table 1.

For the cylinder and wind turbine wakes, we define a fundamental frequency, f_0 , which is used to normalise the frequency axes. This normalisation simplifies the identification of harmonic frequencies, allowing us to refer to them by their integer multiples of f_0 ,

Case	Method	§	Re	f_0	Δt	$N_x \times N_y (\times N_z)$	N_t	N_f	N_{ovlp}	N_{blk}
Cylinder wake	DNS	3.1	100	0.167	0.34	230×101	600	150	75	7
Wind turbine	TR-PIV	3.2	4×10^4	1.87	0.01	769×196	5456	321	161	33
Isotropic turbulence	DNS	3.3	418		0.002	64^3	5024	2512	1884	5

Table 1. Overview of datasets and spectral estimation parameters. The cylinder wake, wind turbine and isotropic turbulence datasets are sourced from Chu & Schmidt (2023), Biswas & Buxton (2024a,b) and Perlman *et al.* (2007) and Li *et al.* (2008), respectively. The fundamental frequency, f_0 , is not applicable to the isotropic turbulence case.

thus making the bispectral plots easier to interpret. For the cylinder wake, the vortex-shedding frequency is used as the fundamental frequency, and for the wind turbine wake, we use the rotor rotational frequency. For the first example, we systematically present the method's primary outcomes and illustrate the interpretation of the results. The second example provides a comprehensive analysis, focusing on the physical explanation of a complex technical flow. The dynamical behaviours of both the cylinder and turbine wakes are dominated by their respective harmonic frequencies. The analysis of forced isotropic turbulence extends TOD to a classical broadband flow, whose dynamics does not exhibit sparsity in the frequency domain.

3.1. Cylinder wake

We start with the cylinder wake, a classic example of a canonical unsteady flow characterised by a well-understood nonlinear dynamics and commonly used in reduced-order modelling (Deane *et al.* 1991; Ma & Karniadakis 2002; Noack *et al.* 2003; Jin *et al.* 2021). Specifically, we analyse the flow around a cylinder at a Reynolds number of $Re = 100$, based on the cylinder diameter and free-stream velocity. The direct numerical simulation (DNS) data for this case is obtained using the polyharmonic splines with polynomial augmentation radial basis function-finite differences implementation of the fractional-step, staggered-grid incompressible Navier–Stokes solver developed by Chu & Schmidt (2023). The cylinder is positioned at the origin with a diameter of 1. The time step in the simulation is adjusted to ensure that a Fourier block of size $N_f = 150$ captures multiple complete flow cycles. The fundamental vortex-shedding frequency is $St_0 = 0.1673$ (Williamson 1988; Barkley 2006; Jiang & Cheng 2017). The flow fields are interpolated onto a Cartesian mesh within the computational domain defined as $x, y \in [-5, 29.35] \times [-5, 5]$. For TOD analysis, velocity gradients are evaluated using fourth-order central finite differences. As demonstrated in Appendix A, the results are insensitive to both the grid spacing and the order of accuracy of the discretisation scheme. In Appendix B, we assess the statistical convergence of our results by repeating the TOD computation using half of the available snapshots, confirming the statistical robustness of TOD.

The first outcomes of TOD are two bispectral plots: one depicting the mode bispectrum and the other illustrating the modal energy budget. These plots provide a comprehensive overview of the triadic phase coupling and energy transfer across frequencies, respectively, and are presented in § 3.1.1. The second outcome consists of three physically interpretable fields: the recipient modes, convective modes and transfer fields. These fields unveil the coherent spatial structures that underpin triadic interactions, highlighting the regions where spectral momentum and kinetic energy are transferred among triads. Examples of these fields and their interpretation are detailed in § 3.1.2.

The mode bispectrum, along with the recipient and convective mode pairs, is computed via the SVD of the two-point cross-bispectral covariance matrix in (2.10). The modal

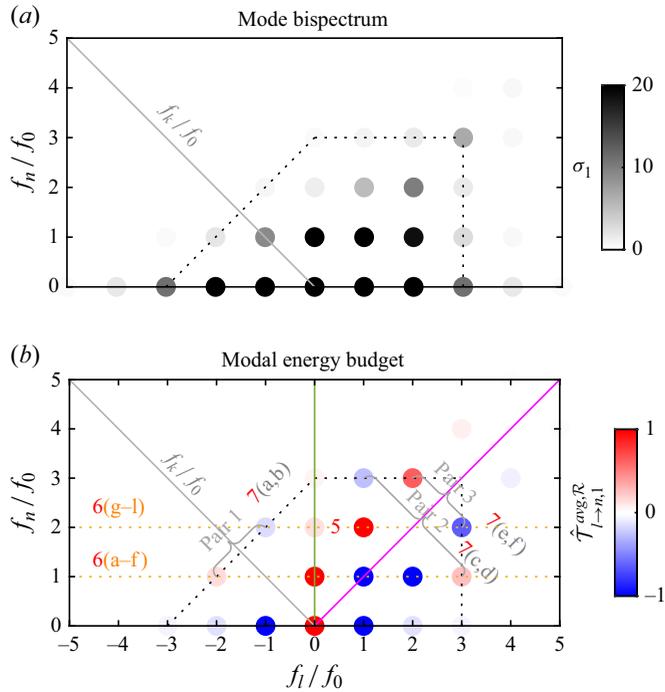


Figure 4. Scatter plots of the leading mode bispectrum (a) and modal energy budget (b) of the cylinder wake at $Re = 100$. Coloured lines mark linear advection (magenta), mean production (green) and the catalyst frequency axis (grey); see figure 3(a). Energy-conserving pairs involving the fundamental vortex-shedding frequency, f_0 , are also marked. Numbers and letters indicate figures depicting the corresponding modes.

energy budget, representing the expected total energy transfer in the bispectral plane, and the corresponding transfer fields are computed from the modes according to definitions (2.51) and (2.54), respectively. In both examples in §§ 3.1–3.2, we show that focusing on the leading, or optimal, donor–recipient pair is sufficient.

3.1.1. Mode bispectrum and modal energy budget

To identify the most active triadic interactions, we present the mode bispectrum, which shows the optimal convective–recipient covariance σ_1 , and the corresponding (real part of the) modal energy transfer $\hat{T}_{l \rightarrow n, 1}$, in figures 4(a) and 4(b), respectively. In both panels, local maxima occur at integer multiples of the fundamental vortex-shedding frequency, underscoring the importance of these harmonics. Triads with negligible amplitudes are omitted for clarity. Each panel emphasises different aspects of the triadic interactions and should be analysed together for a more comprehensive understanding. The mode bispectrum effectively identifies the triads with the strongest phase coupling but does not provide a quantitative measure of the total energy transfer associated with these triads. Conversely, the modal energy budget quantifies these energy transfers but does not directly indicate the strength of the phase couplings. Interestingly, some triads prominent in the bispectrum, such as $(f_i, f_n)/f_0 = (-1, 1)$ and $(2, 2)$, show negligible integral modal energy transfer, suggesting that strong phase coupling can occur alongside zero net energy exchange between the convective modes and the recipient modes in these triads. We highlight that even in the absence of net energy transfer, significant spatial redistribution of kinetic energy may occur, see, e.g. figure 6(k). Therefore, the mode

bispectrum and the energy-budget bispectrum should be interpreted together to provide a complete representation of the triadic interactions.

The most significant triads are observed along the lines $f_n = 0$ and $f_n = f_0$ in figure 4(a). The former indicates that the spatial deformation of the mean flow field is primarily driven by the self-interactions between the harmonics and their corresponding conjugates. Meanwhile, the triads along the line $f_n = f_0$ emphasise the triadic activity at the fundamental vortex-shedding frequency. Additionally, the advection of harmonics is evident along the line $l = n$. Other significant triads, such as the sum interaction of the fundamental instability with itself, contributing to the second harmonic (i.e. $(f_l, f_n)/f_0 = (1, 2)$), are also well captured.

The residual mode bispectrum, which contains all higher modes (not shown), lacks such structure and is negligible compared with the optimal convective–recipient covariance σ_1 , with a covariance fraction of $\sum_{j=2}^7 (\sum_n \sum_l \sigma_j) / \sum_{j=1}^7 (\sum_n \sum_l \sigma_j) \approx 10^{-8}$. This stark separation of singular values indicates that it is sufficient to focus solely on the leading singular value.

Figure 4(b) provides a detailed analysis of the modal energy budget and its directional characteristics for each triad. As expected, the largest energy transport is observed for the self-advection at the origin, $(f_l, f_n)/f_0 = (0, 0)$. Additionally, several active triads with significant energy transfer are found along the line $f_n = f_0$, where the recipient frequency matches the fundamental frequency (lower orange dotted line). It is evident that this frequency gains energy from the mean flow through production and subsequently contributes to the superharmonic via a forward energy cascade. At the triads $(f_l, f_n)/f_0 = (-2, 1)$ and $(3, 1)$, we observe energy being transferred from higher to lower frequencies. While this phenomenon has been termed an inverse energy cascade (Jin *et al.* 2021), we argue that it is more accurately described as a backscatter, because the two triads are isolated in the modal energy budget. By contrast, the triads $(f_l, f_n)/f_0 = \{(0, 1), (1, 2), (2, 3), (3, 4)\}$ form a clear sequence of forward energy transfers that can be interpreted as a cascade.

Interestingly, a negative energy budget is present at $(f_l, f_n)/f_0 = (1, 1)$, indicating an attenuation of vortex shedding when advected by the mean flow. This phenomenon is further illustrated in figure 6(d): a weak energy increment region appears on the centreline downstream of the cylinder, while the overall transfer field is dominated by two symmetrically positioned, strong energy attenuation lobes along the shear layers. While energy attenuation through self-advection by the mean flow is clearly observed within the cylinder wake, no such effect is evident in the wind turbine wake, as illustrated later in figure 10.

When the recipient frequency is the superharmonic (i.e. $f_n/f_0 = 2$), it mainly gains energy from the fundamental frequency and contributes to the third harmonic, demonstrating a successive transfer of energy from lower to higher frequencies. A clear forward energy cascade is apparent in the sequence of positive energy transfers above the $f_n = f_l$ line, as well as the negative energy transfers below it with matching magnitudes. Furthermore, the modal energy budget is anti-symmetric about the $f_n = f_l$ line in the right quadrant, implying that the amount of energy transferred from the donor to the recipient equals the amount received by the recipient. This confirms that there is no net energy gain or loss during the transfer process. In the left quadrant, the anti-symmetry follows the line $f_l + f_n = 0$. These observations exemplify the property of pairwise conservation described in § 2.2.2 and illustrated in figure 3.

As an example, we highlight three easily identifiable conservation pairs located on the same hexagon. Setting $l = 2$ and $n = 3$ in (2.49) shows that these three conservation pairs

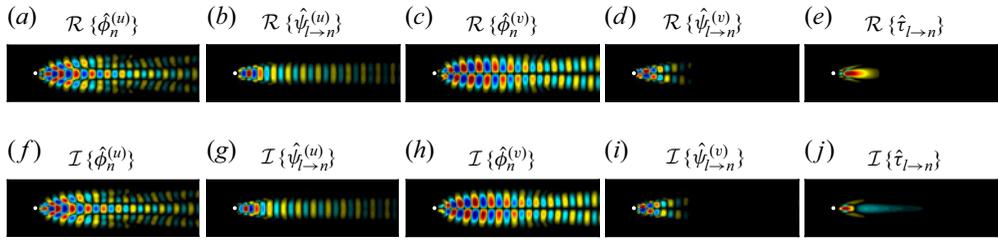


Figure 5. The leading recipient modes, convective modes and transfer fields for the donor–recipient pair $(f_l, f_n)/f_0 = (1, 2)$: (a–e) real parts; (f–j) imaginary parts; (a,b,f,g) streamwise components; (c,d,h,i) transversal components.

constitute the six-triad conserved set

$$\underbrace{\left(\hat{\tau}_{2 \rightarrow 3}^{\mathcal{R}} + \hat{\tau}_{3 \rightarrow 2}^{\mathcal{R}}\right)}_{\text{pair 3}} + \underbrace{\left(\hat{\tau}_{1 \rightarrow 3}^{\mathcal{R}} + \hat{\tau}_{3 \rightarrow 1}^{\mathcal{R}}\right)}_{\text{pair 2}} + \underbrace{\left(\hat{\tau}_{-1 \rightarrow 2}^{\mathcal{R}} + \hat{\tau}_{-2 \rightarrow 1}^{\mathcal{R}}\right)}_{\text{pair 1}} = 0, \quad (3.1)$$

highlighting how both individual pairs and the six-triad set maintain a zero net energy balance. For $l = 0$ and $n = 3$, we obtain the degenerate set

$$\underbrace{\left(\hat{\tau}_{0 \rightarrow 3}^{\mathcal{R}} + \hat{\tau}_{3 \rightarrow 0}^{\mathcal{R}}\right)}_{\text{pair 3}} + \underbrace{\left(\hat{\tau}_{3 \rightarrow 3}^{\mathcal{R}} + \hat{\tau}_{3 \rightarrow 3}^{\mathcal{R}}\right)}_{\substack{\text{degenerate pair} \\ \text{(mean advection)}}} + \underbrace{\left(\hat{\tau}_{0 \rightarrow 3}^{\mathcal{R}} + \hat{\tau}_{-3 \rightarrow 0}^{\mathcal{R}}\right)}_{\text{pair 1}} = 0, \quad (3.2)$$

which contains two repeated triads, $\hat{\tau}_{3 \rightarrow 3}^{\mathcal{R}}$ and $\hat{\tau}_{0 \rightarrow 3}^{\mathcal{R}}$, and therefore only four unique triads. Unlike the triad associated with the advection of the fundamental, $\hat{\tau}_{1 \rightarrow 1}^{\mathcal{R}}$, the triad associated with the advection of the third harmonic, $\hat{\tau}_{3 \rightarrow 3}^{\mathcal{R}}$, has near-zero net energy transfer. This leaves nine triads involving the mean and the first three harmonics with non-zero net energy transfer out of the two sets described by (3.1) and (3.2). When taken together, these nine triads complete the conservation of energy on the hexagon indicated by the dotted line.

3.1.2. Recipient modes, convective modes and transfer fields

In addition to the mode bispectrum and the modal energy budget – both evaluated in an integral sense – TOD analysis provides physically interpretable fields, offering deeper insights into the spatial characteristics of triadic interactions. As a representative example, we visualise the recipient modes, convective modes and transfer fields for $(f_l, f_n)/f_0 = (1, 2)$ in figure 5. The real and imaginary components of the recipient and convective modes display identical flow patterns with a constant streamwise phase shift of approximately $\pi/2$ radians, as expected for non-dispersive structures that simply convect downstream. As noted in § 2.1.2, both fields evolve coherently in time at the recipient frequency. The real component of the transfer field shows that an oval region downstream of the cylinder predominantly gains energy. Therefore, in the following analysis, we will focus on the real components of these fields and the streamwise components of the recipient and convective modes.

Figure 6 presents the leading modes for two recipient frequencies: the harmonic and superharmonic. The leading recipient modes remain consistent across different donor frequencies, while the convective modes vary for the same recipient frequency. Together, these observations emphasise the collective manner in which triads interact to produce the observed flow structures. It is noteworthy that the transfer fields exhibit distinct compact

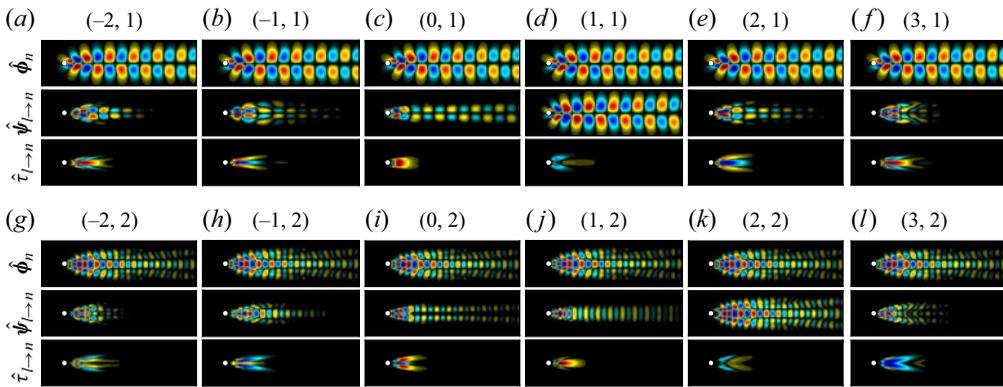


Figure 6. The real part of the leading streamwise recipient and convective modes, and transfer fields of the cylinder wake for the same recipient frequencies: (a–f) $f_n/f_0 = 1$, harmonic; (g–l) $f_n/f_0 = 2$, superharmonic.

support, even though the convective and recipient modes are active far downstream of the cylinder, extending beyond the computational domain. This suggests that the most critical triadic interactions occur close to the cylinder, but manifest with varying spatial structures and, consequently, lead to different physical interpretations for each triad. The transfer field associated with mean production for the first harmonic, $(f_l, f_n)/f_0 = (0, 1)$, is dominated by a compact region of positive energy transfer downstream of the cylinder, which aligns well with findings from experimental structural perturbation analysis (Strykowski & Sreenivasan 1990) and prediction-based wavemaker analysis (Giannetti & Luchini 2007; Marquet, Sipp & Jacquin 2008). The latter, a linear approach, identifies regions in the flow with the strongest localised feedback where dominant instability mechanisms are active. Going beyond linear analysis, the transfer fields now identify these regions as the nonlinearly most active. Almost all transfer fields exhibit mixed spatial patterns of positive and negative energy exchange, reflecting the complexity of the nonlinear interaction mechanisms even in the canonical cylinder wake. Two major energy transfer patterns emerge from interactions involving the harmonic or superharmonic: (i) centreline energy increment accompanied by two symmetrically positioned energy attenuation lobes, as in $(f_l, f_n)/f_0 = \{(-2, 1), (1, 1), (3, 1), (-1, 2)\}$, or (ii) the inverse configuration, as in $(f_l, f_n)/f_0 = \{(-1, 1), (2, 1), (0, 2)\}$. While Jin *et al.* (2021) reported similar patterns when considering the overall nonlinear interactions of the harmonic and subharmonic, TOD uniquely traces these coherent structures back to individual inter-scale transfers. The consistent opposition of these patterns suggests a competitive energy transfer process within specific triads, where the shear layers and the centreline receive opposing contributions. Whereas the self-advection of the harmonic is associated with a negative net energy transfer, that of the superharmonic yields a net transfer of zero. The superharmonic’s distinct, adjacent butterfly-shaped structures of alternating sign indicate that while there is no net energy gain or loss, a process of spatial energy redistribution is occurring, transporting kinetic energy to a more downstream region.

Figure 7 illustrates the leading fields for the three conservative pairs highlighted in figure 4(b) and (3.1). Each pair of transfer fields reveals two distinct structures, but their integral sum remains zero, demonstrating the balanced contributions from both components. This confirms that energy is spatially redistributed through nonlinear interactions within each pair, while the total energy remains conserved. Thus, analysing the entire transfer field structures, rather than just their integral sums, is crucial for

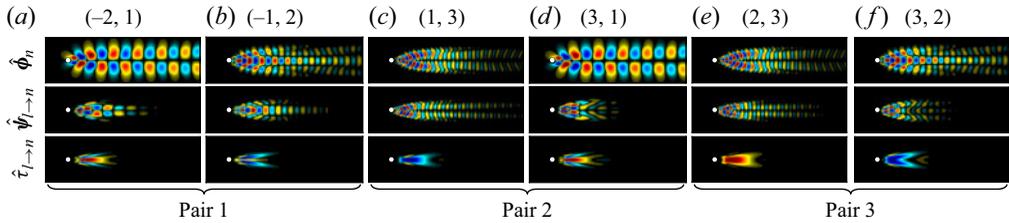


Figure 7. Same as figure 6 but for the three conserved pairs shown in figure 4(b) and (3.1).

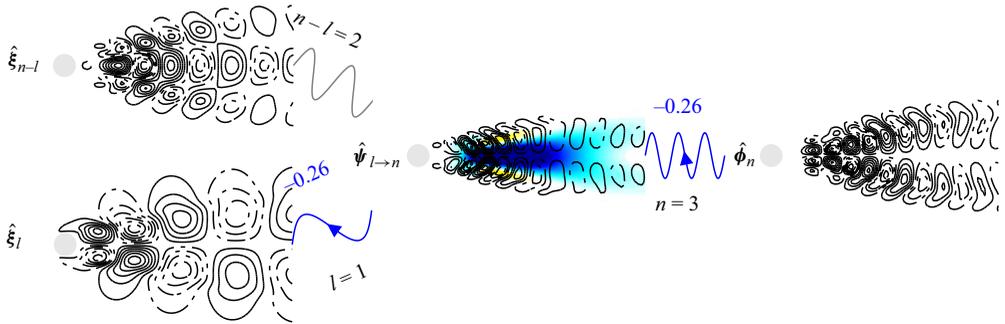


Figure 8. Leading catalyst mode, $\hat{\xi}_{n-1}$, donor mode, $\hat{\xi}_l$, convective mode, $\hat{\psi}_{l \to n}$, and recipient mode, $\hat{\phi}_n$, of the cylinder wake for the (1, 3) triad. For visual clarity, only the subdomain $x, y \in [-2, 10] \times [-2.5, 2.5]$ is displayed. Solid and dash-dotted contour lines correspond to positive and negative fluctuations, respectively. The normalised leading modal energy transfer field, $\hat{\tau}_{l \to n}$, is overlaid, with red and blue corresponding to positive and negative transfers. The integral modal transfer is $\hat{\tau}_{l \to n}^{avg, \mathcal{R}} = -0.26$. This backscatter is represented by the blue arrows pointing from the recipient to the donor.

fully understanding the physical mechanisms underlying these nonlinear interactions. Pairs 1 and 2 display the flow fields associated with the energy backscatter. Interestingly, the energy transfer fields show a similar pattern when the recipient frequencies are lower than the donor frequency but differ significantly when the recipient frequency is higher. This suggests that similar mechanisms govern the energy transfer from the negative superharmonic and the third harmonic back to the fundamental frequency. The magnitudes of the (3, 1) and (-2, 1) energy transfer fields are shown in figure 9 below, illustrating the differences in their respective contributions to the overall transfer.

To demonstrate the TOD modes in the analysis of spectral momentum and energy transfers, figure 8 shows a schematic of the interactions among the catalyst, donor, convective and recipient modes for a representative triad, $(f_l, f_n)/f_0 = (3, 1)$. This triad exemplifies the case of energy backscatter (Jin *et al.* 2021), in which the centreline region of the convective mode is markedly attenuated by the recipient mode. Although the shear-layer region of the convective mode experiences slight energy increments, the net effect is a backscatter of energy from the recipient at higher frequencies to the fundamental donor via triadic interactions. This backscatter reflects the expected long-time-average energy transfer; transient forward transfers are not directly captured. The catalyst mode participates in the interaction but does not explicitly gain or lose energy. Collectively, these TOD modes reveal the coherent spatial structures involved in the triadic interactions and underpin the physical mechanisms driving energy and momentum exchanges.

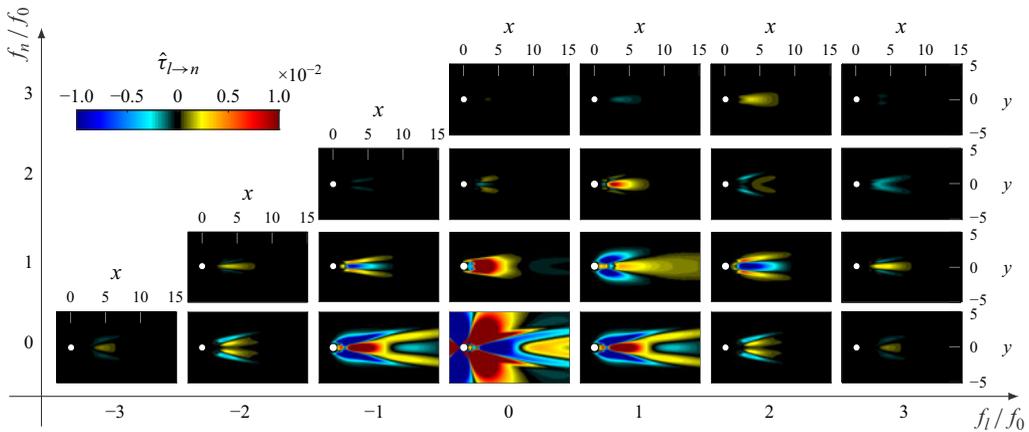


Figure 9. Modal transfer fields between harmonic components in the cylinder wake.

Figure 9 shows the unnormalised transfer fields between harmonics, allowing a direct comparison of the contributions from different donors. Owing to the decay of spectral densities, transfers involving higher-order harmonics exhibit comparatively lower magnitudes. This behaviour is clearly captured in the modal bispectrum presented in figure 4(a). Integrating each transfer field yields the modal energy-budget bispectrum shown in figure 4(b), thereby preserving pairwise conservation. This conservation is not directly evident in the visualisation due to the structural complexity of the transfer fields.

3.2. Wind turbine wake

In the second application, we demonstrate the utility of the new orthogonal decomposition for the identification of dominant triads in experimental data of turbulent flows. We choose the example of the turbulent wake of a model wind turbine from Biswas & Buxton (2024a,b), with a Reynolds number of 40 000 based on the rotor diameter and free-stream velocity. The turbine wake flow exemplifies real-world data of mixed broadband–tonal turbulent flows, whose complex dynamics exhibits both stochastic and deterministic behaviours, in the presence of noise and other measurement artefacts. The data were captured using time-resolved particle image velocimetry (TR-PIV), and correspond to experiment 1A in Biswas & Buxton (2024b). The field of view of the PIV is aligned with the axis of the turbine tower, and spans $x/D \in [0.5, 5]$ and $y/D \in [-0.35, 0.75]$ in the streamwise and transverse directions, respectively, where D is the diameter of the rotor. The rotor revolves at a tip speed ratio of $\lambda = \pi St_0 \approx 6$, where $St_0 = f_0 D / U_\infty = 1.87$ is the rotational frequency and U_∞ is the free-stream velocity. The time series consists of 5456 snapshots separated by a constant spacing of $\Delta t U_\infty / D = 0.01$, and spans 102 revolutions. The period of each revolution is given by $T U_\infty / D = 1 / St_0 = 0.535$. The three-bladed rotor also imposes a blade-passing frequency of $3St_0 = 5.61$ on the flow. Biswas & Buxton (2024b) previously analysed this flow using the technique of optimal mode decomposition (OMD; Wynn *et al.* 2013), itself an extension to dynamic mode decomposition (DMD; Schmid 2010). They extracted coherent structures using OMD, then projected the kinetic energy-budget equation onto the dominant structures, finding significant energy transfer among St_0 , $3St_0$ and their harmonic peaks via triadic interactions. The turbine wake is thus ideal for assessing TOD, which, distinct from OMD and DMD, directly and optimally accounts for triads.

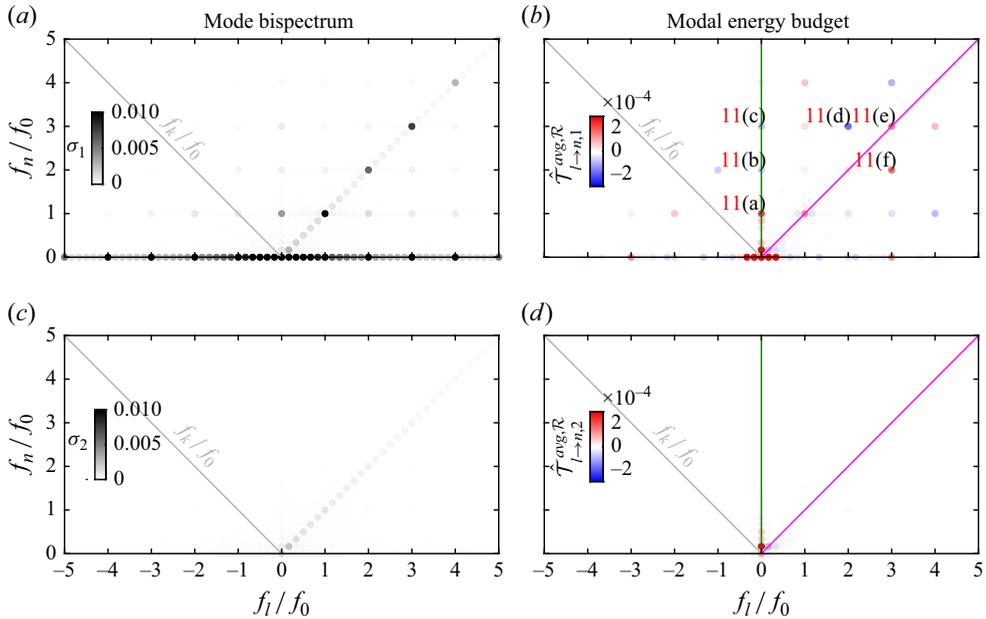


Figure 10. Mode bispectra (a,c) and modal energy budgets (b,d) of the turbine wake: (a,b) leading modes; (c,d) first suboptimal modes.

For spectral estimation, we choose a block size of $N_f = 321$, equivalent to six rotational periods, and use a rectangular window (see table 1 for details). As with the cylinder wake in § 3.1, this combination of parameters ensures that the discrete spectral energy distribution of the turbine wake is accurately captured. Velocity gradients are evaluated using fourth-order central finite differences. Confirmation of statistical convergence is reported in Appendix B.

Figure 10 shows the mode bispectra and modal energy budgets corresponding to the first and second TOD modes. The leading singular values in figure 10(a), summed over the bispectrum plane, account for a convective–recipient covariance fraction of $(\sum_n \sum_l \sigma_1) / (\sum_{j=1}^{N_b} \sum_n \sum_l \sigma_j) = 63\%$, thus confirming the large separation between the leading and suboptimal covariances in the turbulent turbine wake in a global sense. Locally, at the dominant triads, there is a comparable degree of rank separation. For $(f_l, f_n)/f_0 = (0, 1)$ and $(1, 2)$, for instance, the ratio $\sigma_1 / \sum_{j=1}^{N_b} \sigma_j = 76\%$ and 51% , respectively. Away from these dominant triads, the rank separation tends to be very low. For example, $\sigma_1 / \sum_{j=1}^{N_b} \sigma_j = 7\%$ for the triad $(f_l, f_n)/f_0 = (0.5, 1.5)$. By comparison with the optimal singular values, the first suboptimal singular values in figure 10(c) nearly all display substantially reduced magnitudes, except at non-harmonic frequencies along $f_n = f_l$. This is indicative of negligible nonlinear interactions in the suboptimal modes. The integral modal energy budgets in figures 10(b) and 10(d) for the first and second modes, respectively, bolster this claim. Whereas the leading mode exhibits significant transfer of energy over the entirety of the bispectrum, including mean production ($f_l = 0$), linear advection ($f_n = f_l$) and nonlinear transfer, the suboptimal mode shows only mean production at low frequencies, and no other sources of significant energy transport. The TOD modes are thus an efficient representation of the triadic linear and nonlinear dynamics of the flow. In particular, the optimal modes encompass a large portion of the physical phenomena of significance. This is attributable to the approximately periodic

nature of the vortical motions in the turbine wake, and need not hold true in general. A counterexample will be presented in § 3.3.

The leading mode bispectrum and integral modal energy budget reveal a grid-like pattern of local maxima, which identify triads made up of the rotational frequency, f_0 , and its higher harmonics. Unlike the cylinder wake, in the turbine wake, the optimal convective–recipient covariance, σ_1 , and the optimal real integral modal energy transfer, $\hat{\mathcal{T}}_{l \rightarrow n, 1}^{avg, \mathcal{R}}$, both display finite values at non-harmonic frequencies, which reflects the turbulent nature of the flow. However, the magnitudes of both statistics at non-harmonic frequencies tend to be small, so we focus on the harmonics. Because the turbine wake exhibits large singular value separation, we expect – and confirm – that the leading modes of these dominant triads approximately recover the conservation principles described in § 2.2.2 and exemplified by the laminar cylinder wake in § 3.1. With the exception of triads along the MKE line, $f_n = 0$, mean production line, $f_l = 0$, and linear advection line, $f_n = f_l$, the modal energy budget largely obeys pairwise conservation, thus facilitating the interpretation of energy flow from f_l to f_n previously introduced in § 2.2.3.

In the leading mode bispectrum in figure 10(a), the most dominant triads are found along the $f_n = 0$ axis. Each of these triads corresponds to the convective term $\hat{c}_{l \rightarrow 0}$ within the RANS (2.4). They thus conspire to deform the turbulent mean flow through the action of the Reynolds stress, $(\mathbf{u}' \cdot \nabla) \mathbf{u}'$. The diagonal line representing $f_n = f_l$, or equivalently, $f_k = 0$, also displays active triad interactions. Along this line, the local maxima correspond to the term $\hat{c}_{n \rightarrow n}$ in the spectral momentum (2.3), with f_n equal to the harmonic frequencies. These triads arise due to advection of the turbine blade tip vortices by the mean flow. The dominance of these triads in the mode bispectrum reflects the convective nature of the turbine wake. The most active of these triads are $(f_l, f_n)/f_0 = (1, 1)$ and $(3, 3)$. They couple the mean flow to the exogenous forcing, i.e. the rotational and blade-passing frequencies, f_0 and $3f_0$, respectively. Nonlinear triads, which do not involve the mean flow as one frequency component, are also clearly identified in the mode bispectrum, but are considerably weaker than those that do.

The leading integral modal energy budget in figure 10(b) is generally in good agreement with the energy budget and directivity reported by Biswas & Buxton (2024b) based on OMD modes. By recognising mean production and linear advection as special cases of triadic energy transfer, the modal budget enables us to directly compare the importance of these linear mechanisms with the nonlinear transfer between specific donor–recipient frequency pairs. The rotational frequency, $f_n/f_0 = 1$, receives most of its energy from $f_l = 0$ via mean production, $(f_l, f_n)/f_0 = (0, 1)$. The second harmonic, $f_n/f_0 = 2$, receives most of its energy from the blade-passing frequency, $f_l/f_0 = 3$, via the nonlinear transfer $(f_l, f_n)/f_0 = (3, 2)$. The blade-passing frequency, in turn, acts mainly as a donor of energy back to the second harmonic via $(f_l, f_n)/f_0 = (2, 3)$. In Biswas & Buxton (2024b), it was shown that linear advection is the main contribution to the energy budget of $f_n/f_0 = 3$, as the net nonlinear transfer to this frequency is comparatively small. Here, we see that the individual nonlinear transfer, $(f_l, f_n)/f_0 = (2, 3)$, in fact dominates. In other words, on a granular level, the overall spectral energy budget at $f_n/f_0 = 3$ is strongly dependent on nonlinear interactions.

It should come as no surprise that the modal energy budget of the cylinder wake differs significantly from that of the turbine wake. Whereas the former is characterised by the natural instability of vortex shedding, the latter is forced exogenously by the rotating blades. In the turbine wake, at every harmonic frequency, $f_n/f_0 = 1, 2, \dots$, linear advection makes a positive contribution to the energy budget. This observation stands in contrast to the budget of the cylinder wake in figure 4, where linear advection removes

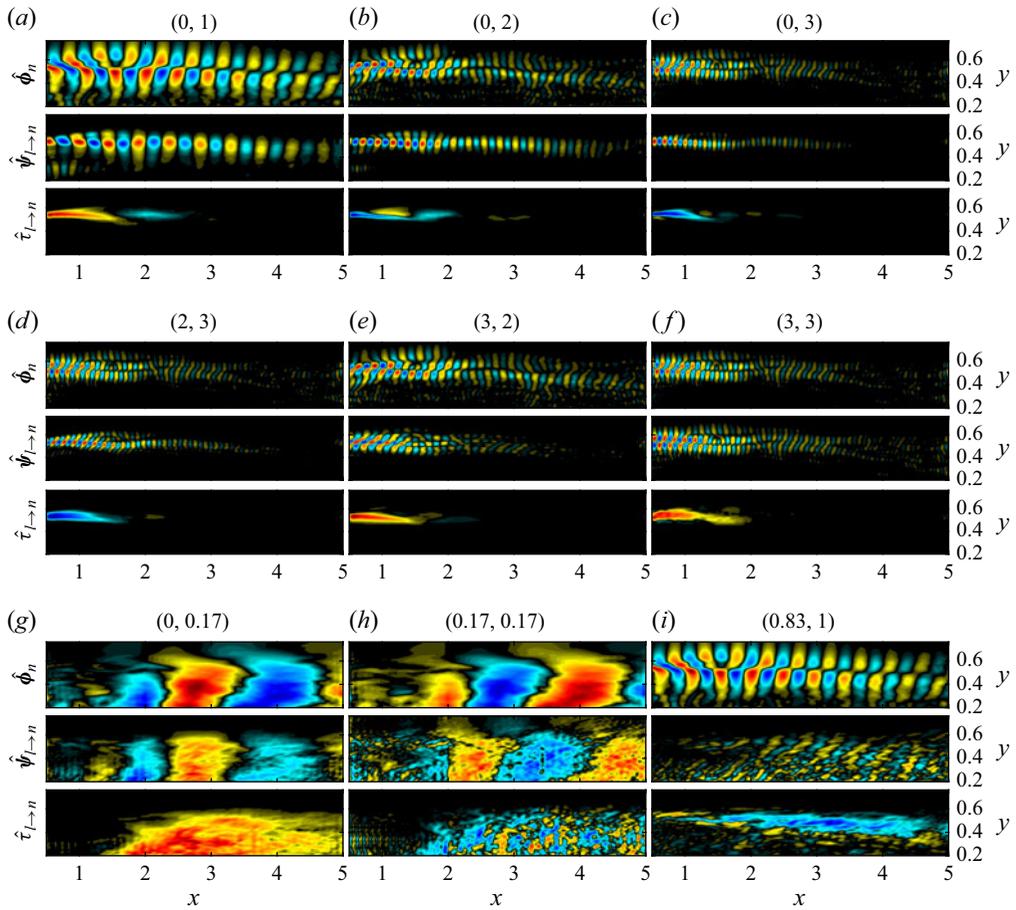


Figure 11. The real part of the leading streamwise recipient modes (first, fourth and seventh rows), streamwise convective modes (second, fifth and eighth rows) and transfer fields (third, sixth and ninth rows) of the turbine wake for different donor–recipient pairs, $(f_l/f_0, f_n/f_0)$: (a) (0, 1); (b) (0, 2); (c) (0, 3); (d) (2, 3); (e) (3, 2); (f) (3, 3); (g) (0, 0.17); (h) (0.17, 0.17); (i) (0.83, 1). Normalised fields $(q/\max\{|q|\}) \in [-1, 1]$ are shown.

energy from $f_n/f_0 = 1$, and preserves the energy of the higher harmonics. Furthermore, in the cylinder wake, mean production injects energy into all harmonics; contributions to $f_n/f_0 = 2, 3$ and higher harmonics are modest but exclusively positive. In the turbine wake, on the other hand, the mean flow contributes energy to $f_n/f_0 = 1$ while extracting energy from $f_n/f_0 = 2, 3$. More generally, nonlinear interactions play a greater role in the overall dynamics of the cylinder wake than they do in the turbine wake.

Figure 11(a–c) shows the leading recipient modes, $\hat{\phi}_{n,1}$, convective modes, $\hat{\psi}_{l \rightarrow n,1}$, and transfer fields, $\hat{\tau}_{l \rightarrow n,1}$, of the three mean production triads, $(f_l, f_n)/f_0 = (0, 1)$, $(0, 2)$ and $(0, 3)$. As an aid to interpretability, to remove experimental artefacts present in the data that are exacerbated by the numerical gradient calculations, we apply a low-pass filter to the streamwise wavenumbers of all modes and transfer fields. In figure 11(a), the streamwise velocity components of the recipient and convective modes for $(f_l, f_n)/f_0 = (0, 1)$ show tip-vortex structures that are deformed by the tip shear layer. The vortices experience slow spatial decay, extending past the end of the PIV window at five rotor diameters from the turbine. The transfer field instead has compact support, and is localised mainly in the

region $x/D \lesssim 3$. The shape of the transfer field is nearly identical to the mean production term computed by Biswas & Buxton (2024b). While the integral modal transfer of this triad is positive, the spatial modal transfer field reveals that the mean flow donates energy to the rotational frequency, $f_n/f_0 = 1$, only in the initial shear layer, $x/D \lesssim 1.5$. This is followed by a smaller, inverse contribution for $x/D \gtrsim 1.5$, within which energy is extracted from $f_n/f_0 = 1$. Biswas & Buxton (2024b) explained the sign change at $x/D \approx 1.5$ in terms of the changing orientation of the vortex triplet shed from the three-bladed rotor, which results in a sign change in the Reynolds stress term, specifically the streamwise and transverse velocity covariance, $\hat{u}\hat{v}$. The transfer field, $\hat{\tau}_{l \rightarrow n, 1}$, which forms a low-rank reconstruction of the production, captures this effect. The streamwise component of the recipient mode, $\hat{\phi}_{n, 1}^{(u)}$, optimally represents \hat{u} . The corresponding component of the convective mode, $\hat{\psi}_{l \rightarrow n, 1}^{(u)}$, optimally represents $\hat{u}\partial\bar{u}/\partial x + \hat{v}\partial\bar{u}/\partial y$. A change in the inclination of the recipient mode, relative to the convective mode, therefore alters the sign of the velocity correlations, $\hat{u}\hat{u}$ and $\hat{u}\hat{v}$, and hence the direction of energy transfer. Figure 11(a) clearly displays this behaviour. For $x/D \lesssim 1.5$, the recipient mode is oriented with the convective mode. At $x/D \approx 1.5$, the convective mode roughly maintains its angle, while the recipient mode abruptly reorients itself against the convective mode.

The influence of the relative orientation of the modes on energy transfer appears robust at the higher harmonics. For the triad $(f_l, f_n)/f_0 = (0, 2)$ in figure 11(b), the convective mode once again approximately maintains its inclination throughout the domain. The recipient mode, meanwhile, reorients itself at $x/D \approx 2.3$, which coincides with the location where the transfer field switches from negative to positive. For $(f_l, f_n)/f_0 = (0, 3)$, the modes are more spatially compact, and the effect is subtle. However, the recipient mode appears to undergo orientation change twice, first at $x/D \approx 1.7$, and again at $x/D \approx 2.3$. Both locations roughly correspond to sign changes in the transfer field.

Figure 11(d–f) shows the modes and transfer fields of the triads $(f_l, f_n)/f_0 = (2, 3)$, $(3, 2)$ and $(3, 3)$. The triads $(2, 3)$ and $(3, 2)$ form a conserved pair in the integral modal energy budget. By way of vortex pairing (Biswas & Buxton 2024b), they transfer energy nonlinearly from $f/f_0 = 3$ to $f/f_0 = 2$. The backscatter of energy towards the second harmonic, which is not exogenously forced, explains its prominence in the mode bispectrum in figure 10(a). In general, the transfer fields of a conserved triad pair need not have the same spatial distribution (with the opposite sign): the energy transfer is conserved only in an integral sense; see (2.48). In this particular case, however, the transfer fields of $(2, 3)$ and $(3, 2)$ appear nearly identical in shape. The recipient and convective modes of triad $(2, 3)$ both evolve at the frequency $f_n/f_0 = 3$. As a consequence of the dispersion relation of the tip vortices, which couple the temporal and spatial frequencies of a wave, the recipient and convective modes also share the same streamwise wavenumber. This is also true of the modes of triad $(3, 2)$. The recipient and convective modes oscillate at the frequency $f_n/f_0 = 2$, and have a lower wavenumber than the modes of triad $(2, 3)$. For the mean production triads in figure 11(a–c), only the recipient modes are tilted, while the convective modes are roughly aligned with the mean flow. In contrast, for triads $(2, 3)$ and $(3, 2)$, both recipient and convective modes manifest pronounced tilt. Using (2.36) and (2.30), catalyst and donor modes associated with each triad can be individually visualised. An example is given in figure 12 for the triad $(f_l, f_n)/f_0 = (3, 2)$. The catalyst mode, convective and recipient modes, and donor mode range respectively over three harmonic frequencies, $f_{n-l} = -1$, $f_n = 2$ and $f_l = 3$. We observe that the dominant wavenumber of each mode also scales hierarchically, with the higher-frequency modes possessing larger streamwise and transverse wavenumbers. Superposing the convective mode on the modal transfer field highlights their distinct spatial support: whereas the convective mode has

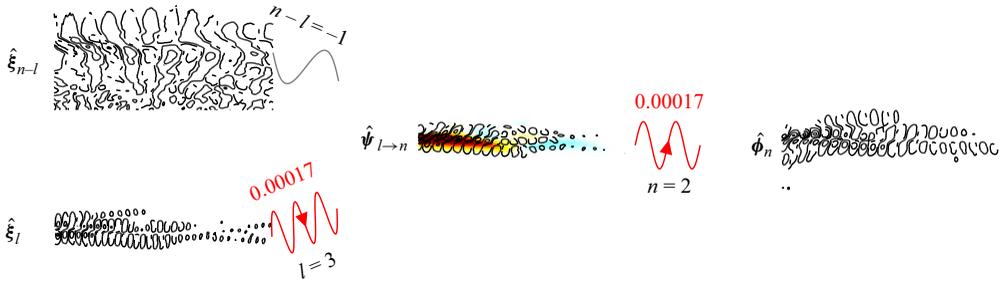


Figure 12. Leading catalyst mode, $\hat{\xi}_{n-l}$, donor mode, $\hat{\xi}_l$, convective mode, $\hat{\psi}_{l \rightarrow n}$, and recipient mode, $\hat{\phi}_n$, of the turbine wake for the (3, 2) triad. The subdomain $x \leq 3$ is displayed. Solid and dash-dotted contour lines correspond to positive and negative fluctuations, respectively. The normalised leading modal energy transfer field, $\hat{\tau}_{l \rightarrow n}$, is overlaid, with red and blue corresponding to positive and negative transfers. The integral modal transfer is a forward transfer of $\hat{\tau}_{l \rightarrow n}^{avg, \mathcal{R}} = 0.00017$ and is represented by the red arrows pointing from donor to recipient.

finite amplitude outside the shear layer, the modal transfer field is radially confined within the shear layer.

In addition to tip-vortex modes, low-frequency oscillations attributed to wake meandering (Biswas & Buxton 2024b) are also captured by the leading mode bispectrum and modal energy budget in figure 10(a,b). Triadic interactions involving these wake meandering modes are primarily found in the vicinity of the origin, $(f_i, f_n)/f_0 = (0, 0)$, as well as near (1, 1). The optimal recipient modes, convective modes and modal transfer fields of three such triads, (0, 0.17), (0.17, 0.17) and (0.83, 1), are reported in figure 11(g–i). The length of the dataset limits an accurate determination of the frequency of these structures as well as complete mode convergence, so in the following we focus on qualitatively interpreting the physics. The triads (0, 0.17) and (0.17, 0.17) represent the linear mechanisms of production and advection, respectively. Their recipient and convective modes are all structurally similar, revealing large-scale waves with a streamwise wavelength of approximately $2D$. The waves are inclined with respect to the rotor plane and oriented with the mean shear. The corresponding modal transfer fields, unlike those associated with the tip-vortex modes in figure 11(a–f), are spatially extended in both the streamwise and transverse directions and supported far downstream of the rotors at $x = 0$. Because the recipient and convective modes maintain a consistent orientation throughout the domain, there is no clear sign change in the transfer fields. This enables significant energy transfer, particularly for the (0, 0.17) triad, which draws energy from the mean flow and deposits it into the meandering wake. The production peaks in the region $2 \lesssim x/D \lesssim 3$. In contrast, the mean advection triad, (0.17, 0.17), is responsible for dampening the wake meandering.

The nonlinear triad (0.83, 1), in which the low-frequency wave acts as the catalyst, possesses a comparatively low convective–recipient covariance in the leading mode bispectrum. Previously, Biswas & Buxton (2024b) found that the wake meandering mode transfers only a small amount of energy via nonlinear interaction when the transfer is integrated over the domain. As we have emphasised, in general the integral energy transfer is vulnerable to the possibility of spatial cancellation, i.e. a small global transfer could mask large local transfers. The convective–recipient covariance, not being subject to cancellation, is a more direct and robust measure of the strength of triadic interactions. The low covariance of the (0.83, 1) triad indicates that the interaction itself is weak.

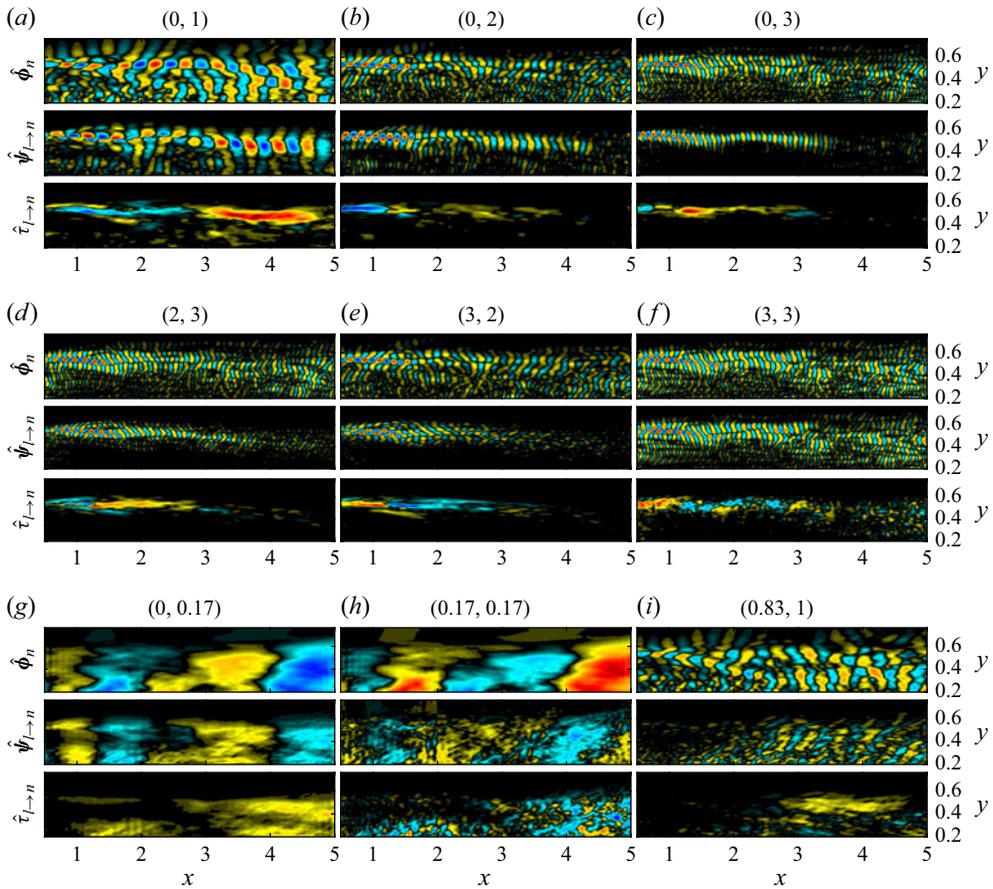


Figure 13. Same as figure 11 but for the first suboptimal modes.

By virtue of the bounds in (2.52), the weak interaction then guarantees a small integral energy transfer. The recipient mode of this triad (figure 11*i*) displays the typical tip-vortex structure. The corresponding convective mode shows waves oriented with the mean shear and active primarily downstream of $x/D \approx 1$, as expected from the spatial support of the catalytic wake meandering. These waves also differ from the tip-vortex convective modes in terms of their spatial evolution. Specifically, whereas the present convective mode is spatially amplified, the convective modes in figure 11(*a–f*) all experience spatial decay. As a consequence of the spatial growth of the (0.83, 1) convective mode, its corresponding modal transfer field also peaks downstream (in absolute terms), at $x/D \approx 3$.

We also report the first suboptimal TOD modes of the turbine wake in figure 13. The triads included correspond to those in figure 11. Because the suboptimal singular values are small and the interactions weak (see figure 10*c*), fully converging the suboptimal modes would require significantly longer data, and the modes should be viewed with caution. Notwithstanding this caveat, we can draw some qualitative inferences. Comparing figures 11 and 13, the recipient and convective modes associated with different modal ranks contain waves with approximately the same spatial scales, suggesting that they implicitly follow the same dispersion relations. The suboptimal modes have greater streamwise extension and are thus less spatially damped than the optimal modes. This phenomenon

is clearest in the tip-vortex structures in figures 11(a–f) and 13(a–f). Most notably, the suboptimal recipient and convective modes of the (0, 1) triad in figure 13(a) each alternates between spatial decay and growth. The suboptimal convective mode, for instance, is damped for $x \lesssim 2.5$, then amplified for $x \gtrsim 2.5$.

Between ranks one and two, most of the triads preserve the direction of their local energy transfer, that is, the sign of the modal transfer field. This is true of the triads (0, 2), (0, 3) and (2, 3), which for both modes one and two exhibit negative transfer in the near wake, as well as the triad (0.17, 0.17), which removes energy from the meandering waves in the far wake. Similarly, the triads (3, 2) and (3, 3) consistently contribute energy to the near wake, and the triad (0, 0.17) consistently contributes to the far wake. The triads (0, 1) and (0.83, 1) serve as counterexamples, showing inversions of the energy directivity. In the case of (0, 1), the inversion is explained by changes in the tilt of the tip vortices. In the recipient mode in figure 11(a), the vortices are initially tilted against the mean shear for small x , then rotates to align themselves with the mean shear further downstream. By comparison, the vortex tilting is reversed in the recipient mode in figure 13(a). Here, the vortices are initially oriented with the mean shear, before rotating to oppose the mean shear for large x . An analogous mechanism is hinted at by figure 13(i) in the case of the triad (0.83, 1). However, the ill-converged modes cannot be interpreted with confidence. For triads involving only the harmonics of f_0 , i.e. those in figures 11(a–f) and 13(a–f), the suboptimal transfer fields extend to much larger x . The (0, 1) triad, in particular, transfers energy only in the near wake for mode one, but does so in both the near and far wake for mode two. This is in accordance with the differing rates of spatial growth and decay exhibited by the recipient and convective modes. Nonetheless, as the bispectra in figure 10 demonstrate, overall the suboptimal modes make only a minor contribution to the energy transport relative to the leading modes. This is a natural consequence of the low-rank nature of the turbine wake. In § 3.3, we present an application where consideration of the suboptimal modes is essential to fully characterising the flow physics.

3.3. Forced isotropic turbulence

While TOD finds obvious applications in periodic flows and flows with periodic statistics, it need not be confined to such cases. Our third example is the DNS data of forced isotropic turbulence drawn from the Johns Hopkins Turbulence Database (Perlman *et al.* 2007; Li *et al.* 2008). Contrary to the cylinder and turbine wakes in §§ 3.1–3.2, this is an example of a broadband turbulent flow. As the dataset has already been extensively documented, we offer only a brief overview. The simulation was carried out using a pseudo-spectral solver on a triply periodic grid of size 1024^3 that spans the domain $x, y, z \in [0, 2\pi]$. The Taylor-scale Reynolds number is approximately 418. The time series comprises 5024 snapshots of the velocity field over a duration of about five large-eddy turnover times, saved at a time interval of $\Delta t = 0.002$. Since we are interested in the inertial rather than the dissipation range, for computational tractability we downsample the snapshots onto a grid size of $N_x = N_y = N_z = 64$. To reduce spectral leakage, we use a relatively large block size of $N_f = 2512$ and the Hamming window (see table 1 for details about spectral estimation). Velocity gradients are evaluated using Fourier spectral differentiation. Confirmation of statistical convergence is reported in Appendix B.

Owing to the broadband nature of the flow, we present the leading TOD mode bispectrum and modal energy budget, shown respectively in figure 14(a,b), as continuous contours rather than scatter plots. In figure 14(a), the leading mode bispectrum displays convective–recipient covariances that vary smoothly as a function of frequencies f_l and f_n and peak at low frequencies, in agreement with the classical bispectra predicted by

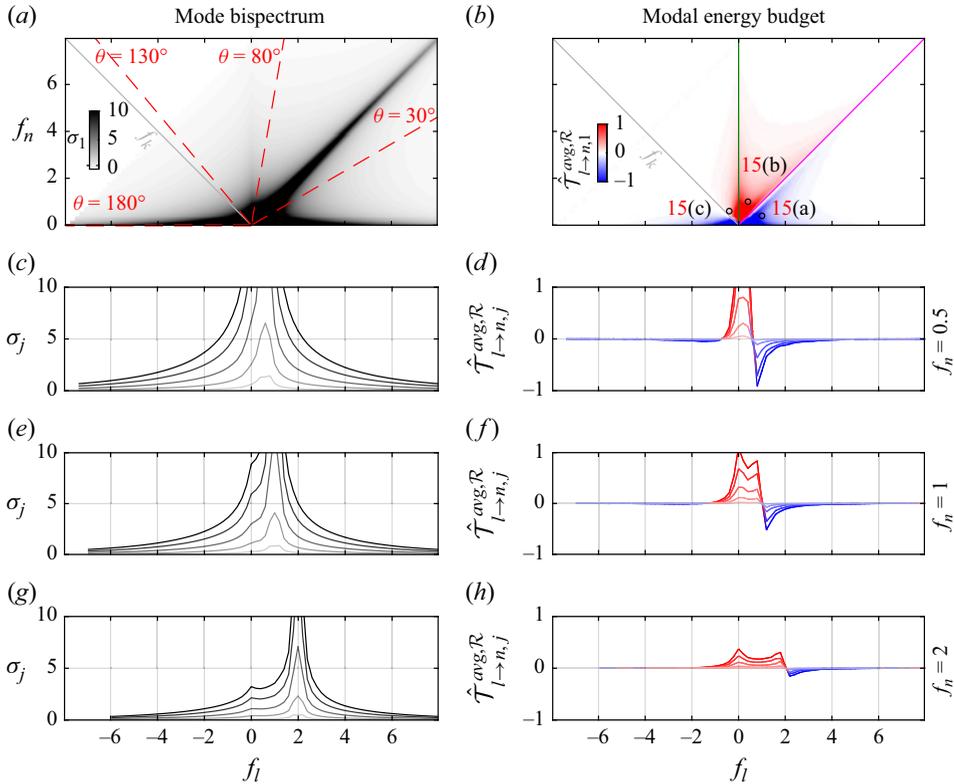


Figure 14. Mode bispectra (*a,c,e,g*) and modal energy budgets (*b,d,f,h*) of the isotropic turbulence. In panel (*a*), red dashed lines correspond to the polar angles, $\theta = \tan^{-1}(f_n/f_i)$, examined in figure 17. In panel (*b*), circled triads correspond to the modes reported in figure 15. Panels (*c–h*) display constant- f_n sections of the bispectra and budgets: (*c,d*) $f_n = 0.5$; (*e,f*) $f_n = 1$; (*g,h*) $f_n = 2$. The shading of the curves varies from dark to light with higher mode numbers, j . The colours in panels (*d,f,h*) indicate positive (red) or negative (blue) transfers.

Herring (1980) for point signals. For the cylinder and wind turbine wakes, a clear grid pattern emerges in their mode bispectra due to correlation between the tonal components of those flows. For the isotropic turbulence, as anticipated, no such peaks appear. In other words, the nonlinear dynamics is not amenable to a sparse representation by only a small subset of triads. The exceptions are triads involving the zero-frequency component, $f_i = 0$ (production), $f_n = 0$ (transfer–production difference) or $f_i = f_n$ (linear advection), as well as those in the surrounding region. While these triads appear dominant, they are in fact contaminated by spectral leakage from unresolved low-frequency structures (Lii, Rosenblatt & Van Atta 1976). Following Herring (1980), slices of the optimal and suboptimal mode bispectra along constant f_n are presented in figure 14(*c,e,g*). While the spectra differ in magnitude, qualitatively they follow the same distribution over f_i . Along each fixed f_n , apart from the peaks at $f_i = 0$ and $f_i = f_n$, the spectra decay smoothly with $|f_i|$. The gaps between the mode bispectra of different mode numbers are all similar and relatively small, i.e. the statistics of the nonlinear dynamics is not low-rank.

Representative examples of leading recipient modes, convective modes and modal transfer fields for the triads $(f_i, f_n) = (1, 0.4)$, $(-0.4, 0.6)$ and $(0.4, 1)$ are reported in figure 15(*a,b,c*), respectively. These triads are members of the same conserved set (see figure 3). The modes are arranged in ascending order of recipient frequency,

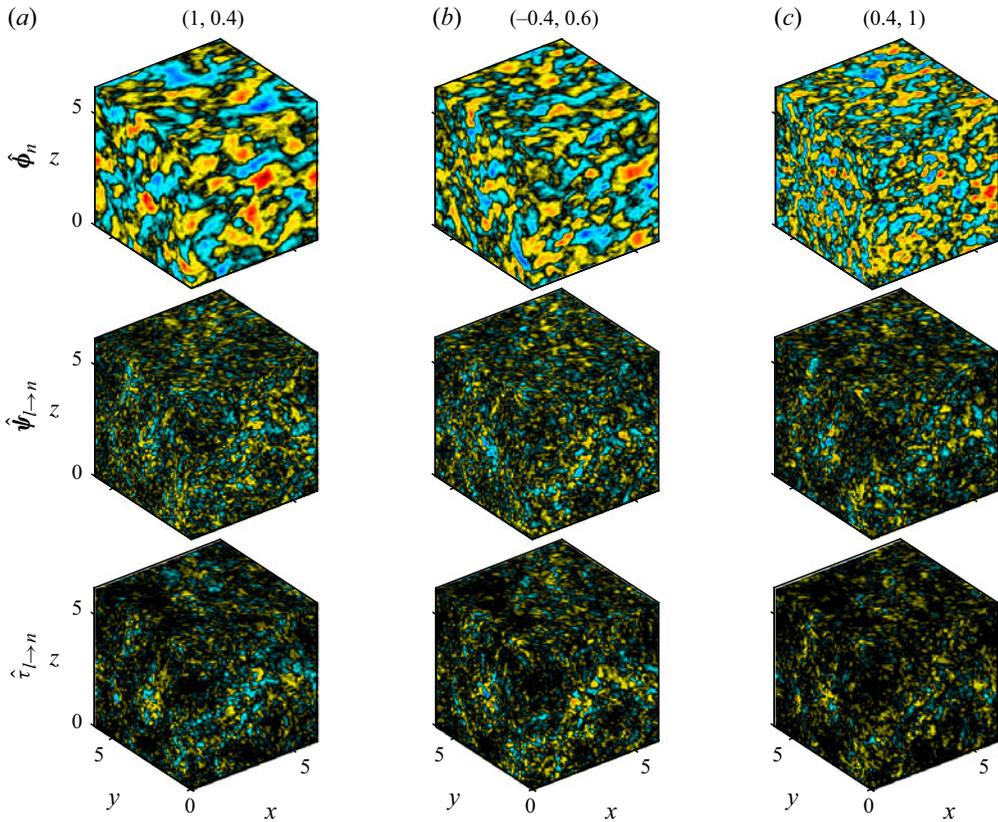


Figure 15. The real part of the optimal streamwise recipient modes (first row), streamwise convective modes (second row) and transfer fields (third row) of the isotropic turbulence for different donor–recipient pairs, (f_l, f_n) : (a) (1, 0.4); (b) (−0.4, 0.6); (c) (0.4, 1). These triads are highlighted in figure 14(b).

f_n . Accordingly, the recipient modes, $\hat{\phi}_n$, display the expected increase in average wavenumber with frequency. For the (0.4, 1) triad, the catalyst and donor modes on the $x = 0$ plane are reported in figure 16 along with the convective and recipient modes and modal transfer field. Consistent with (2.54), which relates the modes and transfer field, the regions in the convective mode where the velocity fluctuations are highest overlap with the regions of intense local energy transfer. The modes and modal transfer fields in figures 15 and 16 show evidence of inadequate statistical convergence. Given the statistical homogeneities of the turbulence, in the limit of infinite data we expect the mode shapes to converge to sinusoids (Sirovich 1987; Holmes *et al.* 2012). However, like all spectral modal decompositions, TOD requires a careful balance between convergence and frequency resolution. More pertinently, by design TOD excels at identifying spatio-temporally persistent structures and coherent dynamics, as measured by the convective–recipient covariance (2.8). In contrast, triadic interactions in isotropic turbulence are intermittent (Batchelor & Townsend 1949) in space and time, challenging any technique that seeks global motions sustained over long time horizons. Within flows such as isotropic turbulence, TOD inherently highlights long-lived interactions, while de-emphasising transient bursts of momentum and energy transfer. If the goal is to educe short-lived structures associated with localised, transient interactions, TOD may be augmented by defining a finite space–time horizon conditioned on the most intense energy exchanges (Hack & Schmidt 2021; Park & Lozano-Durán 2025). However, this exceeds the scope of

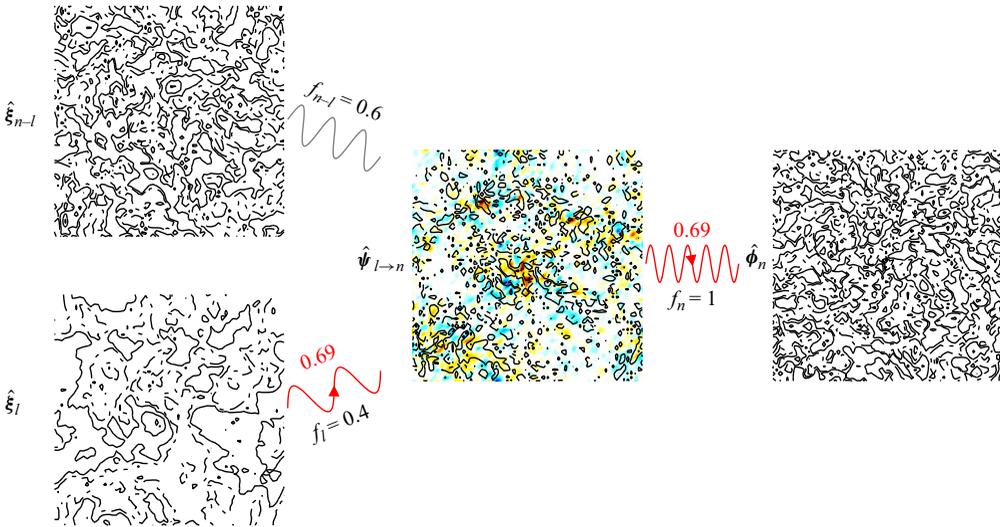


Figure 16. Leading catalyst mode, $\hat{\xi}_{n-l}$, donor mode, $\hat{\xi}_l$, convective mode, $\hat{\psi}_{l \to n}$, and recipient mode, $\hat{\phi}_n$, of the isotropic turbulence case for the (0.4, 1) triad. The $x = 0$ plane is shown. Solid and dash-dotted contour lines correspond to positive and negative fluctuations, respectively. The normalised leading modal energy transfer field, $\hat{\tau}_{l \to n}$, is overlaid, with red and blue corresponding to positive and negative transfers. The integral modal transfer is a forward transfer of $\hat{\mathcal{T}}_{l \to n}^{avg, \mathcal{R}} = 0.69$ and is represented by the red arrows pointing from donor to recipient.

our present study. In view of these constraints, in the remainder of this section we focus our analysis on the mode bispectra and modal energy budgets. Although the modes are challenging to converge, the mode bispectra and budgets, which are integral statistics, demonstrate rapid convergence.

It is well known that, at sufficiently high Reynolds numbers, in the inertial subrange the turbulent statistics gains independence from the viscosity (Kolmogorov 1941). For isotropic turbulence, Van Atta (1979) and Herring (1980) independently derived a model of the classical bispectrum on the basis of dimensional analysis and the test field model (Kraichnan 1971a), respectively. Related ideas were earlier put forward by von Kármán (1948). In accordance with Kolmogorov’s second hypothesis, the model, obtained for spatial Fourier components, depends only on the dissipation rate, ϵ , and the triad wavenumbers. Following Van Atta (1979)’s dimensional arguments, we extend this model to the frequency space. We model the convective–recipient covariance using the ansatz

$$\sigma_j(f_l, f_n) = \epsilon f^m H_j(\theta), \quad (3.3)$$

where $\epsilon = 0.103$ (obtained from the dataset; see Li *et al.* 2008), $f^2 = f_l^2 + f_n^2$ and $\theta = \tan^{-1}(f_n/f_l)$ parameterise the donor and recipient frequencies in polar coordinates, H_j is an unknown dimensionless function and m is an undetermined constant. From its definition (2.8), the convective–recipient covariance has the dimensions of length squared over time, thus requiring $m = -2$. Figure 17 displays σ_j for several representative polar angles, θ . The dissipation range is not included in the TOD calculations. The frequency ranges of the spectra differ from each other because of the finite bin size in the bispectrum plane. For $\theta = 30^\circ$, 80° and 130° , the slope of the optimal and suboptimal singular values adheres approximately to the theoretical f^{-2} scaling. An exception is $\theta = 30^\circ$ in figure 17(a), where the spectra, particularly for large j , begin to flatten below $f \approx 0.7$. The polar angle $\theta = 180^\circ$ in figure 17(d), however, entirely departs from the theoretical

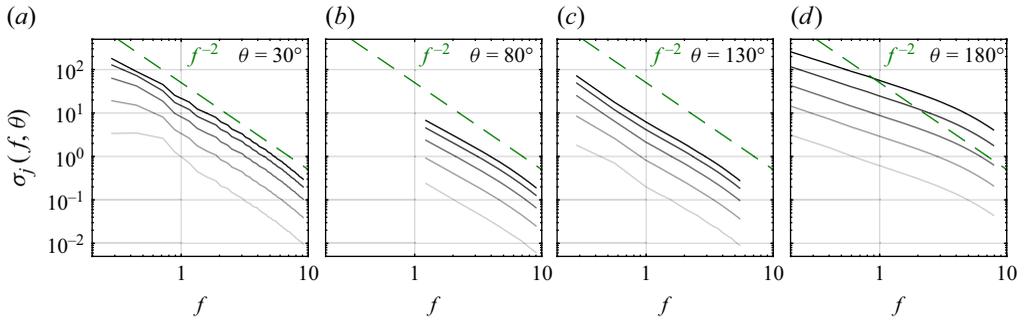


Figure 17. Mode bispectra of the isotropic turbulence in polar coordinates, $f^2 = f_l^2 + f_n^2$ and $\theta = \tan^{-1}(f_n/f_l)$, at constant θ : (a) $\theta = 30^\circ$; (b) $\theta = 80^\circ$; (c) $\theta = 130^\circ$; (d) $\theta = 180^\circ$. The shading of the curves varies from black to grey with higher mode numbers, j . Green dashed lines (---) mark the theoretical frequency scaling of the mode bispectra, $\sigma_j \propto f^{-2}$.

scaling. Rather, it appears to scale as f^{-1} . Triads at or near this angle correspond to $f_n \approx 0$ and consequently suffer from spectral leakage. As is clear from the mode bispectrum in figure 14(a), $\theta \approx 0^\circ, 45^\circ$ and 90° are similarly contaminated. The same issue was addressed in the work of Lii *et al.* (1976) by considering velocity derivatives as a form of prewhitening. Excluding these angles, however, for the frequencies of interest the simple model in (3.3) offers a good description of the exponential decay of the singular values.

Whereas the leading mode bispectrum is relatively evenly distributed over all triads, the leading modal energy budget in figure 14(b) is strongly dependent on frequency. The positive and negative energy transfers in the regions $f_n > |f_l|$ and $f_n < |f_l|$, respectively, are clear indications (see the schema in figure 3f) of the classic forward energy cascade from low to high frequencies, and hence from large to small eddies (Richardson 1922; Pope 2000). The modal transfers are approximately conserved with respect to the lines $f_n = \pm f_l$. For example, the conserved triad pair in figure 15(a,c), which straddles the linear advection line, $f_n = f_l$, exchanges energy in the amounts $\hat{\mathcal{T}}_{1 \rightarrow 0.4,1}^{avg, \mathcal{R}} = -0.60$ and $\hat{\mathcal{T}}_{0.4 \rightarrow 1,1}^{avg, \mathcal{R}} = 0.69$, respectively. Here, for notational compactness the subscript $(\cdot) \rightarrow (\cdot)$ designates frequencies rather than frequency indices. The slight discrepancy between the two modal transfers is to be expected, because the conservation property in (2.48) is strictly satisfied only by the full flow field, rather than by an individual modal rank. The transfer vanishes along $f_n = f_l$, as shown in figure 14(b) and exemplified in figure 14(d,f,h). As well as being a by-product of triad conservation, the vanishing transfer along the linear advection line makes sense on obvious physical grounds: the flow is non-advective. In § 2.2.3, we surmised that conservation of the integral modal energy transfer, $\hat{\mathcal{T}}_{l \rightarrow n, j}^{avg}$, might only hold if the nonlinear dynamics is low-rank, because the leading modal transfer would approximate the total transfer, $\hat{\mathcal{T}}_{l \rightarrow n}^{avg}$. Remarkably, the isotropic turbulence example demonstrates empirically that even in flows that are far from low-rank, the conservation of modal transfer remains robust.

The modal budget also displays a clear preference for energy exchanges involving positive donor frequencies, $f_l > 0$. By comparison, negative donor frequencies transfer energy only sparingly, at least in an integral sense. In general, suboptimal TOD modes need not exhibit the same energy dynamics as the leading modes, as the wind turbine example in § 3.2 demonstrates. In this case, however, the modal energy transfers differ among optimal and suboptimal modes in quantity only, while the direction is preserved.

Furthermore, the preference for $f_l > 0$ extends to the suboptimal modes. The constant- f_n slices in figure 14(d,f,h) bear out these behaviours. While triads involving frequency pairs $\pm f$ are sometimes considered in aggregate (Jin *et al.* 2021; Freeman *et al.* 2024; Ding *et al.* 2025), the pronounced imbalance here between $\pm f_l$ suggests positive and negative donor frequencies are deserving of individual attention. We express the preference for positive donor frequencies as

$$\underbrace{\left| \hat{\mathcal{T}}_{l \rightarrow n, j}^{avg, \mathcal{R}} \right|}_{(f_{n-l}, f_l, f_n)} > \underbrace{\left| \hat{\mathcal{T}}_{-l \rightarrow n, j}^{avg, \mathcal{R}} \right|}_{(f_{n+l}, -f_l, f_n)}, \quad (3.4)$$

where, without loss of generality, we use the sign convention $f_l, f_n > 0$. For the sake of clarity, we explicitly write out the catalyst frequency, f_{n-l} or f_{n+l} , as part of each triad. Moreover, it is straightforward to verify that

$$|f_{n+l}| > |f_{n-l}|. \quad (3.5)$$

That is, f_{n+l} represents a higher-frequency – and smaller-scale – structure than f_{n-l} . Noting the reality of the data, the two donor frequencies are mathematically equivalent, $\hat{\mathbf{u}}_l = \hat{\mathbf{u}}_{-l}^*$. We thus attribute the imbalance solely to the differing catalyst frequencies. In words, the inequality (3.4) says that a low-frequency, large-scale structure (i.e. f_{n-l}) is a more efficient catalyst or mediator of scale-to-scale modal energy transfer than a high-frequency, small-scale structure. Our result accords with previous wavenumber-domain investigations (Domaradzki & Rogallo 1990; Ohkitani & Kida 1992), which have shown that among the compatible catalysts for a given donor–recipient pair (Batchelor 1953) the small-magnitude catalyst wavenumbers contribute most to the transfer.

It is also noteworthy that no overt preference for $f_l > 0$ is visible in the convective–recipient covariances in figure 14(a), implying the expansion coefficients of convective and recipient modes are similarly correlated for $\pm f_l$. As (2.51) and (2.52) reveal, the integral modal energy transfer is the covariance scaled by the extent of spatial alignment between the corresponding convective and recipient modes. Consequently, either a large covariance or a close alignment between the mode pair (or both) can lead to a large energy transfer, i.e. energy transfer may be rooted in statistical or geometric origins. Since the covariances for $\pm f_l$ are comparable, we conclude that low-frequency structures more efficiently catalyse energy transfer because they enable greater geometric alignment between convective and recipient mode structures.

The mode bispectra and modal energy budgets also allow us to confront the long-standing debate about the scale-localness of nonlinear interactions from the perspective of spatio-temporal coherent structures. Whereas early models of spectral energy transport in isotropic turbulence (e.g. Kraichnan 1959) were essentially local, i.e. assuming energy flow between neighbouring wavenumbers, these models were challenged by later investigators who offered evidence of non-localness, i.e. transfer between distant wavenumbers (Kraichnan 1971*b*; Deissler 1978, and others). The latter understanding has been variously bolstered or qualified by more recent studies (Domaradzki & Rogallo 1990; Waleffe 1992; Ohkitani & Kida 1992; Zhou 1993*a*; Cardesa *et al.* 2015, 2017; Johnson 2020; Khurshid, Donzis & Sreenivasan 2021; Lozano-Durán & Arranz 2022), with a growing consensus in favour of predominantly local transfer. The seeming simplicity of this question belies ambiguities in how localness ought to be defined and measured (Zhou 1993*b*; Kishida *et al.* 1999; Eyink 2005; Verma *et al.* 2005). The classical bispectrum provides one intuitive measure of localness: triads on or near the diagonals, which couple similar frequencies, are local, while those away from the diagonals are non-local (Lii *et al.* 1976; Herring 1980). Following this interpretation, the leading mode bispectrum and modal energy

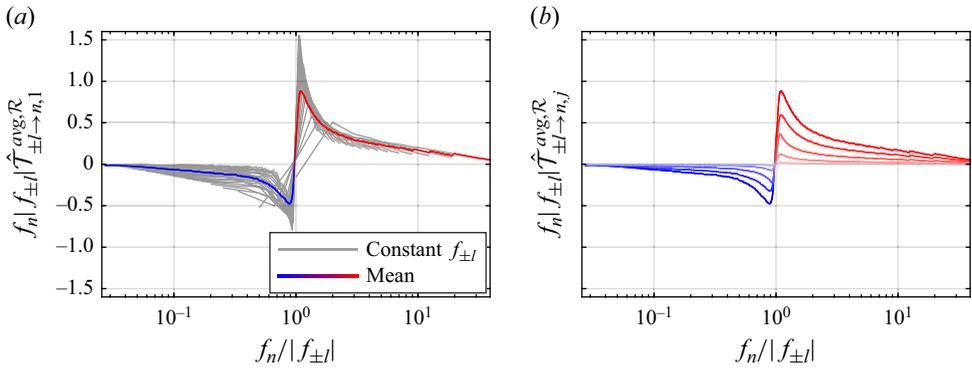


Figure 18. Self-similar-scaled modal energy transfer of the isotropic turbulence: (a) leading mode; (b) all modes. In panel (a), each grey curve corresponds to a fixed $f_{\pm l}$, while the blue–red curve is the mean over all grey curves. Panel (b) displays the mean for each mode number, j , with lighter shading representing higher j . Blue and red segments indicate negative and positive transfers, respectively.

budget in figure 14(a,b) visually support the presence of non-local interactions, as they display finite values away from the diagonals, $f_n = |f_l|$. The large magnitude of the mode bispectrum along $f_n = f_l$ also points to strong local interactions, which is consistent with the Kolmogorov picture of a turbulent cascade, although with the available data we cannot rule out leakage as a competing explanation. As we observed previously, the leading mode bispectrum is more evenly distributed throughout the bispectrum plane than the leading modal budget. The convective–recipient covariance, which is not subject to spatial cancellation, thus suggests a greater prevalence of non-localness than the integral modal energy transfer does. That said, in the context of spatio-temporal data, a core advantage of TOD is that it does offer both perspectives – correlation and energy transfer, and links them through (2.51).

Having confirmed in figure 17 that the mode bispectra conform approximately to inertial-range scaling, we can apply a self-similar scaling to the modal energy transfer (Kraichnan 1967; Zhou 1993b) and quantify the degree of localness in the isotropic turbulence. By summing the contributions to the integral transfer from positive and negative donor frequency pairs

$$\hat{f}_{\pm l \rightarrow n, j}^{avg, \mathcal{R}} = \hat{f}_{l \rightarrow n, j}^{avg, \mathcal{R}} + \hat{f}_{-l \rightarrow n, j}^{avg, \mathcal{R}}, \tag{3.6}$$

then scaling the sum appropriately by donor and recipient frequencies, we obtain the compact representation in figure 18 of the modal energy budget. Figure 18(a) shows the leading mode. Each of the grey curves corresponds to one (absolute) value of the donor frequency, $f_{\pm l}$. The mean over all donor frequencies is marked by the blue–red curve and vanishes when the transfer is precisely local, $f_n/|f_{\pm l}| = 1$, as is consistent with figure 14(b). The transfer is negative (blue) for $f_n/|f_{\pm l}| < 1$ and positive (red) for $f_n/|f_{\pm l}| > 1$, signifying a purely forward cascade. Exact conservation of energy would have yielded a profile that is antisymmetric with respect to $f_n/|f_{\pm l}| = 1$. While perfect antisymmetry is not realised here for reasons already outlined, the overall shape of the profile is correctly captured. Because the energy transfer is strictly self-preserving only if all three frequencies in a triad are confined to the inertial subrange, the collapse of the grey curves is also incomplete. However, the degree of collapse is comparable to that obtained by Zhou (1993b, figure 15), whose simulation benefits from being explicitly constrained to the ideal Kolmogorov inertial-range power spectrum. In the absence of

such constraints, the reasonable self-similar collapse exhibited by TOD exemplifies the advantages of considering global spatio-temporal statistics.

Figure 18(b) repeats the foregoing for the suboptimal modes. Since the quality of the self-similar collapse does not vary significantly with mode number, we report only the average over all donor frequencies. All leading and suboptimal modes behave consistently and reveal key features of the forward cascade. The transfers are maximised in the immediate vicinity of $f_n/|f_i| = 1$, then decay approximately exponentially (with some distortion due to leakage) as f_n diverges from f_i . These results lend further credence to the view that isotropic turbulence is driven predominantly by local energy transfer, with non-local transfer present but playing a subordinate role.

It should not be overlooked that in figure 18(b) all the transfer curves are essentially scaled versions of each other. A novel implication of this is that the spectral transport of energy via nonlinear interactions is not only self-preserving in the full flow field – as is well documented – but also self-preserving within the portion of the flow corresponding to an individual mode number. Given that TOD solves an independent singular value problem for each triad, in general there is no expectation for the decompositions of distinct triads to be related. An analogous, but little remarked, feature also arises in the context of distinct frequencies in SPOD (Schmidt *et al.* 2018). The recovery of inter-triad scaling and self-similarity (figures 17 and 18) at each rank is therefore a particularly encouraging finding, as it is evidence that the decomposition does not merely offer an abstract mathematical formalism, but discovers behaviours that are verifiable and physically meaningful. That the decomposition respects such essential characteristics as self-similarity may also confer benefits on model reduction. Compared with classical Galerkin models that employ triad truncation (Kraichnan 1973; Waleffe 1992; Schmid & Henningson 2001), which may not conserve the flow topology of the full system (Moffatt 2014), a nonlinear reduced model based on rank truncation that retains a broad range of triad interactions but only the low-order TOD modes could yield more success.

4. Discussion and summary

In this work, we have developed a novel modal decomposition which distils coherent flow structures that optimally account for triadic interactions. For a given triad, TOD decomposes the convective and recipient terms in the momentum equations into a pair of jointly optimal orthogonal modal bases, ranked in terms of the covariance between the projections onto the respective bases. Maximising the covariance also captures the bispectral statistics that emerge due to prevalent triadic interactions within the flow. By considering a three-wave interaction as the covariance between the convective and recipient terms, we show that the jointly optimal bases are given by the left and right singular vectors of a singular value decomposition of the convective–recipient covariance tensor. The strength of the correlation is given by the singular value, and measures the statistical prevalence of a given triad. The singular values of all triads are presented in the form of a mode bispectrum, providing a succinct indication of active triads. For non-zero recipient frequencies, $f_n \neq 0$, the convective and recipient mode pair models the inter-scale momentum transfer as governed by the spectral momentum equations. For $f_n = 0$, the mode pair instead contributes to the Reynolds stresses in the RANS equations, thereby acting on the mean flow.

By relating the spectral momentum equations to the spectral kinetic energy equation, we show that a natural consequence of the scale-to-scale modal momentum transfer is the modal energy transfer. The latter is obtained from the singular value and the projection of the convective mode onto the recipient mode. In each triad, the convective and recipient

modes of different orders are uncorrelated and, on average, do not exchange energy with each other. We interpret the spatially integrated modal energy transfer as a modal energy budget. This interpretation follows from recognising that the convective term in the kinetic energy equation can be partitioned into spectral TKE contributions from linear advection by the mean, production by the mean and nonlinear transfer, as well as MKE contributions from mean self-advection, production and nonlinear transfer. The relative importance of each of these budget terms to the overall energy dynamics can be directly read off the bispectrum plane. Specifically, MKE budget terms reside on the $f_n = 0$ axis, while spectral TKE budget terms are active off that axis. Within the TKE budget, linear advection by the mean resides on the $f_n = f_l$ diagonal, and production by the mean is found on the $f_l = 0$ axis.

As required by the globally conservative nature of the convective term, the nonlinear energy transfer of individual triads, summed over the bispectrum plane, vanishes. Perhaps lesser known is that triadic energy transfer is conserved also in a pairwise fashion. Since TOD modes form an optimal representation of the recipient and convective fields, under the condition of large separation between the optimal and suboptimal convective–recipient covariances, the integral modal energy transfer is also pairwise conservative. Taking advantage of pairwise conservation, we use the modal energy budget to systematically excavate and quantify the energy donated from an arbitrary frequency, f_l , and received by another arbitrary frequency, f_n , mediated by a catalyst frequency, $f_k = f_n - f_l$. Positive energy transfers (marked red in this work) in the region $f_n > |f_l|$, or equivalently, negative energy transfers (marked blue) in the region $f_n < |f_l|$, indicate a forward cascade of energy from low to high frequencies. Conversely, negative transfers in $f_n > |f_l|$, or equivalently, positive transfers in $f_n < |f_l|$, signify an inverse cascade of energy from high to low frequencies.

It must be emphasised that finite correlation between the recipient and convective terms does not imply a causal association between the two. That is, the TOD mode bispectrum and modal energy transfer establish a statistical and energetic link between the donor, recipient and catalyst frequencies, but not a causal link. TOD is therefore not equipped to resolve the questions of which frequency arises first, or which frequency generates which, nor is it obvious that such questions are in fact meaningful. Rather, in this work, a triad interaction is understood as the collective action of all three frequency components that contributes to the momentum and energy dynamics of a flow.

Collectively, the scale-to-scale energy exchanges enabled by all donor–recipient pairs are woven into a network of energy transport. Energy enters and leaves the system through the $f_l = 0$ axis and the $f_n = f_l$ diagonal. These represent respectively production and linear advection by the mean – linear mechanisms that are unrestrained by energy conservation. Once inside the system, energy is scattered over the nonlinear regions of the bispectrum plane, circulating from donor to recipient and bound by a hierarchy of conservative properties.

We applied TOD to three examples.

The first example is DNS data of a laminar unsteady cylinder wake, a benchmark problem for canonical convective flows with a well-understood nonlinear dynamics. The leading TOD mode bispectrum reveals that the most dominant triadic interactions occur at integer multiples of the fundamental frequency, confirming that the intrinsic bluff-body vortex-shedding instability mechanism is primarily driven by interactions between the harmonics and their corresponding conjugates. The energy transfer predominantly cascades forward, moving from lower to higher frequencies and thus from larger to smaller spatial scales. This forward energy cascade is evident from the primarily positive energy transfer above the $f_n = f_l$ line and the negative energy transfer below it. Notably, we

also observe triads exhibiting the opposite behaviour, indicative of an inverse energy transfer. The associated energy transfer fields suggest that the mechanism responsible for this backscatter is the pronounced energy attenuation region along the wake centreline. The modal energy transfer fields between harmonic components elucidate the complex interaction mechanisms within individual triads.

Each pair of transfer fields associated with a conserved triad pair reveals two distinct structures whose integral sum is zero, demonstrating the balanced contributions of both components. This indicates that, while energy is spatially redistributed through nonlinear interactions within the computational domain, the total energy is conserved. This finding offers deeper insight, emphasising that a comprehensive analysis of the full structure of the transfer fields, rather than just their integral sums, is essential for fully understanding the physical mechanisms driving these nonlinear interactions.

The second example is TR-PIV data of a turbulent wind turbine wake. This flow is characterised by both deterministic and stochastic components. Moreover, the case is representative of experimental data that are inevitably contaminated by measurement noise. The leading TOD mode bispectrum identifies dominant triads made up of the harmonics of the rotor rotational frequency. Triads with the highest magnitudes are located on the $f_n = 0$ axis, indicating a substantial contribution to the mean flow from the Reynolds stress, and on the $f_n = f_l$ diagonal, indicating interaction between the mean flow and the rotor tip vortices. Overall, these linear triadic interactions are significantly stronger than the nonlinear triads in the mode bispectrum. In the turbine wake, the optimal convective–recipient covariance captures the majority of the total covariance. The nonlinear triads within the leading modal energy budget thus successfully recover the pairwise conservative property. The modal budget provides clear evidence of production, linear advection and nonlinear transfer between the harmonic frequencies. Production by the mean injects energy into the rotational frequency, but extracts energy from its higher harmonics. Linear advection by the mean injects energy into all harmonics. Negative and positive nonlinear transfers above and below the $f_n = f_l$ diagonal, respectively, exemplify an inverse energy cascade. The recipient modes reveal vortex shedding from the rotor tips. Like the cylinder wake, the modal transfer fields have compact support. The streamwise locations of sign changes in the transfer fields are explained in terms of re-orientation of the recipient mode against the convective mode, or *vice versa*. The transfer field corresponding to the inverse transfer from the third to the second harmonic and the transfer field of the conserved pair appear nearly identical in spatial structure. The optimal TOD mode bispectrum also captures triadic interactions between the mean flow, tip-vortex modes and low-frequency wake meandering modes. Their corresponding recipient and convective modes are long-wavelength coherent structures oriented with the mean shear. The spatial distribution of the modal transfer fields explains the large energy transfer from the mean flow to the far wake. While the optimal modes are damped rapidly downstream of the rotor plane, particularly at higher frequencies, the suboptimal modes experience slower spatial decay and may even be amplified after the initial decay. Most, but notably not all, triads transfer energy in the same direction via the optimal and suboptimal modes, i.e. the sign of the energy transfer is generally rank-independent. Reversals in the energy directivity between ranks, when they occur, are attributed to vortex tilting. As is almost always the case with experimental diagnostics, some information is lost when two-dimensional data are captured from a statistically three-dimensional flow, such as the turbine wake. This inherently limits the terms in the momentum equations that can be modelled using TOD. In particular, out-of-plane velocity and velocity gradients, which contribute to momentum and energy transfer, cannot be captured. That the

method nevertheless succeeds in uncovering the nonlinear dynamics of the turbine wake supports its broad applicability to data sets in which the full flow state is only partially observable.

The third example is DNS data of forced isotropic turbulence in a periodic box. Unlike the two previous examples, this is a classical broadband flow that is stochastic and devoid of tonal spectral components. Because of these characteristics, the nonlinear dynamics is not low-rank and instead includes significant contributions from the suboptimal TOD modes, in addition to the optimal. Consistent with the broadband nature of the turbulence, the mode bispectra are smoothly distributed over all frequencies. Using dimensional analysis, we propose and successfully validate an f^{-2} frequency scaling of the convective–recipient covariance over the inertial subrange. The modal energy budgets indicate a purely forward cascade of energy, and conservation of the modal transfer is shown to be robust. For each (absolute) donor–recipient frequency pair, that is, for each $(|f_l|, f_n)$, the modal energy transfer favours the positive over the negative donor frequency. Equivalently, it favours low-frequency over high-frequency catalysts. Through a geometric interpretation of the modal transfer, we establish that low-frequency catalysts effectively align convective modes with recipient modes, leading to efficient energy flow. We also investigate the scale-localness of triadic interactions in the inertial range using TOD, finding a stronger suggestion of non-localness in the mode bispectra than in the modal energy budgets. From an energy view, we quantify the degree of localness via a self-similar transformation of the integral modal transfer. The transformed modal transfer peaks when the recipient and donor frequencies are comparable, but rapidly decays otherwise, supporting the preponderance of local transfer. Strikingly, TOD of isotropic turbulence reveals that each rank of the decomposition is individually self-similar, and not only self-similar in aggregate as observed in prior studies. This exposes the intimate connection between TOD modes and fundamental nonlinear flow physics.

To extract maximal performance from TOD, the following best practices should be adhered to. TOD, much like other frequency-domain methods, benefits greatly from the careful selection of spectral estimation parameters that strikes a good balance between minimising statistical variance and bias. The same recommendations that apply to the standard SPOD algorithm (Towne *et al.* 2018; Schmidt & Colonius 2020) based on Welch’s overlapped segment averaging (WOSA; Welch 1967) should be followed here. For tonal or broadband–tonal flows characterised by some fundamental frequency or period, the need to resolve harmonic peaks accurately places additional constraints on the segment length, $N_f \Delta t$. To centre the peaks in the frequency bins of the DFT and thus avoid spectral leakage, this length should be (close to) an integer multiple of the fundamental period. This renders the data (nearly) periodic within each segment. For periodic data, a rectangular window should be used. More detail on spectral estimation for data with time-periodic statistics can be found in Heidt & Colonius (2024b). For data with particularly challenging demands on spectral resolution and statistical convergence, TOD may benefit from replacing the standard Welch estimator with a multitaper estimator (Thomson 1982; Riedel & Sidorenko 1995) by analogy with multitaper SPOD (Schmidt 2022; Yeung & Schmidt 2024). Other techniques that aim to improve the convergence of SPOD may also be applicable to TOD, including recent algorithms proposed by Blanco *et al.* (2022), Colanera, Schmidt & Chiatto (2025), Heidt & Colonius (2024a) and Yeung & Schmidt (2024).

If primitive variables are not available, as is common with experimental data such as schlieren images, it will not be possible to compute the terms in the momentum equations. For these data sets, we can still apply TOD by forgoing its specialisation to the momentum

and kinetic energy equations. For a general observable, $\mathbf{q}(\mathbf{x}, t)$, we can redefine the cross-bispectral covariance tensor as

$$\mathbf{S}(\mathbf{x}, \mathbf{x}') = E \left\{ \hat{\mathbf{q}}_{(n-l)ol}(\mathbf{x}) \hat{\mathbf{q}}_n^H(\mathbf{x}') \right\} \quad \text{for each } (l, n), \quad (4.1)$$

where $\hat{\mathbf{q}}_{(n-l)ol} \equiv \hat{\mathbf{q}}_{n-l} \circ \hat{\mathbf{q}}_l$. The treatment of nonlinear coupling in (4.1) is closely connected to BMD (Schmidt 2020), which also uses the term $\hat{\mathbf{q}}_{(n-l)ol}$. The SVE of \mathbf{S} yields a mode bispectrum and two sets of modal bases, with the same properties as the algorithm in § 2.1.3. Without access to velocity, the modal energy flow analysis in § 2.2 is not possible. Nevertheless, even in this restricted setting, we have confirmed that TOD succeeds in identifying dominant triads, as triad interactions give rise to correlation among frequency components of the observable, which TOD detects.

Extensions of TOD to study other types of dynamics may be envisioned. By construction, TOD expansion coefficients are uncorrelated; see (2.13). As we saw in § 2.2.3, a direct consequence is that, on average, no energy is exchanged between leading and suboptimal recipient and convective modes. In turbulent flows with a time-periodic mean flow, each leading TOD mode may be interpreted as a deterministic component of the mean, and the suboptimal modes as the stochastic turbulence. The uncorrelatedness between leading and suboptimal modes thus translates to the uncorrelatedness between the deterministic and stochastic components of the flow. In contrast, the findings of Heidt & Colonius (2024b) in relation to CS-SPOD formalises the concept that finite correlation can arise between a periodic mean and the underlying turbulent statistics. Momentum and energy transfers between deterministic motions and the turbulent fluctuations about them are well-trodden territory, and are typically considered by subjecting the equations of motion to a triple decomposition (Hussain & Reynolds 1970), producing coupled equations for the deterministic (organised) and stochastic (turbulent) components (Reynolds & Hussain 1972). Recent applications include Oberleithner, Rukes & Soria (2014) and Heidt *et al.* (2023) for turbulent jets, and Kinjangi & Foti (2023) for a turbulent cylinder wake. For the examples in this work, the large separation between the leading and suboptimal convective–recipient covariances in the TOD mode bispectrum demonstrates that the dynamics of each flow is already well described by only the organised motion. In flows where this is not the case, we can easily investigate organised motion–turbulence interactions using TOD by considering the momentum equations for the turbulent component (Reynolds & Hussain 1972, (2.8)), in place of the full momentum (2.1a). This may be accomplished by removing the periodic mean from the recipient frequency and either the donor or the catalyst frequency of each triad. For example, removing the periodic mean from the recipient and catalyst, while keeping it for the donor, extracts the correlation and transfer between a deterministic–stochastic donor–recipient pair.

In addition to the Navier–Stokes equations, TOD can be generalised to other triadically nonlinear phenomena. For instance, it is straightforward to incorporate velocity and magnetic fields into the recipient and convective terms, then apply TOD to study spectral kinetic and magnetic energy transfers in magnetohydrodynamic turbulence.

The TOD method also opens the door to novel ROMs for which the modal bases, rather than being constructed in a linear framework, are instead tailored to triadic interactions and thus optimal in capturing the convective nonlinearity of the Navier–Stokes equations. As suggested by Schmid & Henningson (2001), the dynamical equations of a ROM should preserve the conservative property of the nonlinear term. They showed that the nonlinear part of truncated equations that use conserved sets of triads – like the ones in § 2.2.2 – as building blocks, indeed conserves energy. For data with a large separation between

the leading and suboptimal TOD singular values, these conservative principles are also satisfied by the leading TOD modes and transfer fields. The TOD method should therefore facilitate the construction of energy-conserving ROMs. When selecting a truncated set of triads to include in a model, it is crucial to consult the mode bispectrum in addition to the modal energy budget. While the modal budget provides copious physical insights, it is the spatial integral of the transfer field and thus prone to cancellation. A triad with small integral energy transfer can have a large singular value (see [figures 4 and 10](#)). In contrast, singular values in the mode bispectrum, which measure covariance, directly convey the statistical significance of each triad.

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Declaration of interests. The authors declare that they have no competing interests.

Data availability statement. A MATLAB implementation of Triadic Orthogonal Decomposition (TOD) is available at <https://github.com/FlowPhysicsGroup/Triadic-Orthogonal-Decomposition>. The data that support the findings of this study are available from the corresponding author upon reasonable request.

Author contributions. B.Y. and T.C. contributed equally to the conceptualisation, formal analysis, methodology, implementation, validation, visualisation and writing of the manuscript. T.C. conducted the cylinder wake example, while B.Y. carried out the wind turbine wake and the isotropic turbulence examples. O.T.S. provided conceptualisation, methodology, project administration, resources, supervision and writing. All authors discussed the results and contributed to the final manuscript.

Appendix A. Spatial discretisation

In TOD, calculating the convective field, $\hat{\mathbf{C}}_{l \rightarrow n}$, requires the evaluation of numerical gradients; see (2.17b). For all the applications we have studied, we find the results to be effectively insensitive to the mesh discretisation and the order of accuracy of the differentiation scheme. As demonstration, we repeat the TOD analyses of the cylinder wake (§ 3.1) and wind turbine wake (§ 3.2), but either doubling the grid spacing ([figure 19](#) left column) or switching from fourth- to second-order central finite difference schemes ([figure 19](#) right column). All results are indistinguishable from those in § 3. We have confirmed that this applies also to the isotropic turbulence case (§ 3.3). Apart from minor visual differences, TOD modes (not shown) are likewise robust. We remind readers that the gradient computation in TOD is merely a post-processing step, and does not require the high accuracies that would be expected of a flow solver, for instance. Nevertheless, other properties of the differentiation scheme, such as numerical dissipation, could have an impact on the results. When applying TOD, the implementation of the gradient operator should therefore always be verified.

Appendix B. Statistical convergence

In this appendix, we verify the statistical convergence of our results in § 3 by repeating each TOD analysis using only the first half of the snapshots in each dataset. For the cylinder and wind turbine wakes, spectral estimation parameters are kept unchanged from §§ 3.1 and 3.2, respectively. For the isotropic turbulence example, the truncated dataset

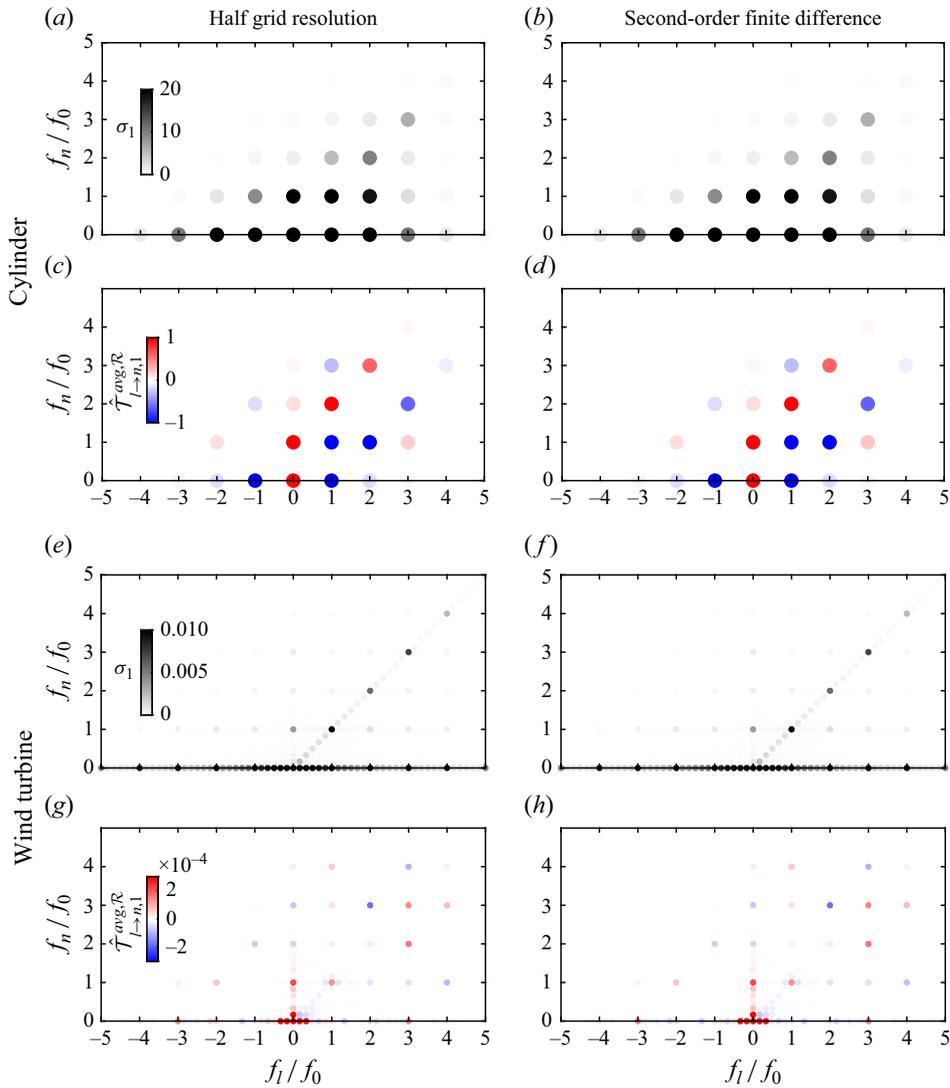


Figure 19. Leading TOD mode bispectra (a,b,e,f) and modal energy budgets (c,d,g,h) of the cylinder wake (a–d) and wind turbine wake (e–h): (a,c,e,g) the grid resolution is halved along each spatial direction compared with § 3, while the finite difference scheme is the same as § 3; (b,d,f,h) the grid resolution is the same as § 3, while the order of accuracy is lowered from fourth to second. The colour scales match those in figure 4 for the cylinder case and figure 10 for the turbine case.

is too short to retain the original block size in § 3.3, so the block size is reduced from $N_f = 2512$ to 1256. The leading TOD mode bispectra and modal energy budgets for the three truncated datasets are reported in figure 20. As expected, the results for the cylinder flow in figure 20(a,b) are identical to those in figure 4 calculated from the full dataset because the flow is deterministic. For the two turbulent flows, minor numerical differences arise between figures 20(c,d) and 10(a,b) in the turbine case, and between figures 20(e,f) and 14(a,b) in the isotropic turbulence case. For instance, in the turbine wake the (1, 0) triad yields a larger modal energy transfer in the truncated data than in the full data. These numerical discrepancies are attributable to intermittency in turbulent flows. Comparison

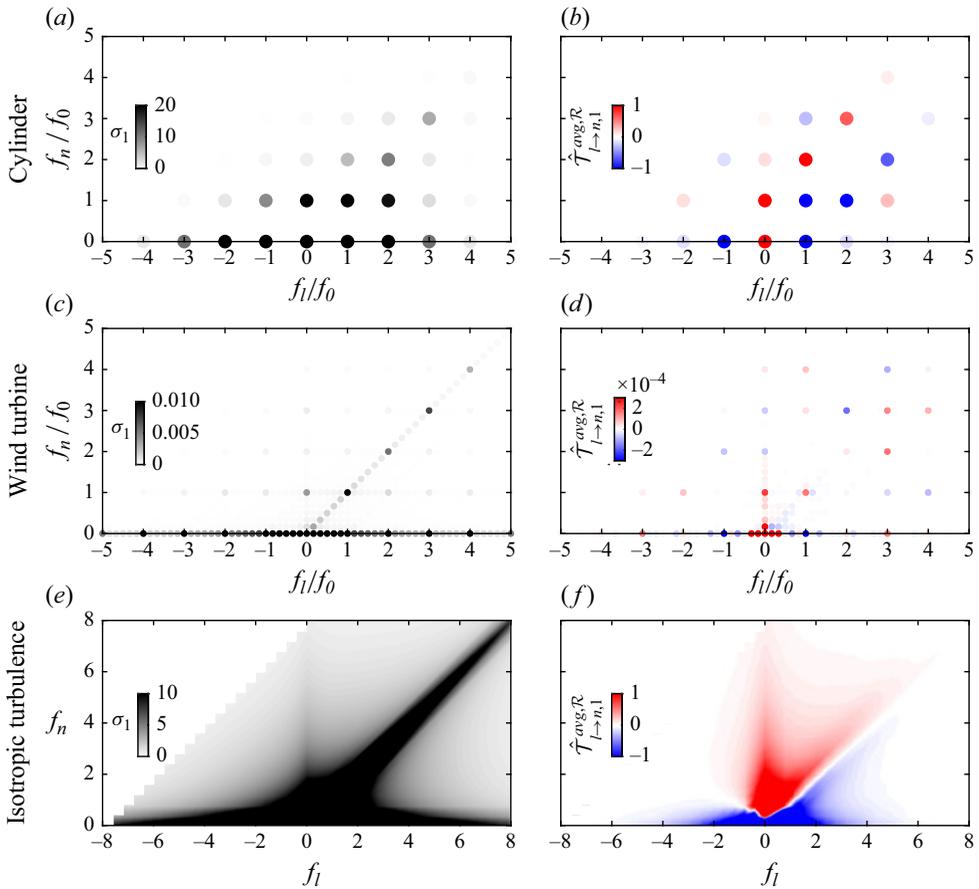


Figure 20. Leading mode bispectra (left column) and modal energy budgets (right column) computed from the first half of each dataset: (a,b) cylinder wake (§ 3.1); (c,d) wind turbine wake (§ 3.2); (e,f) isotropic turbulence (§ 3.3). The colour bars match those in figures 4, 10 and 14.

of the mode bispectra, modal budgets, as well as the modes and transfer fields (not shown) shows no qualitative difference that impacts our main physical findings, indicating the statistical robustness of TOD.

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